

ARCHIVES

11-13-82

33443

P-11

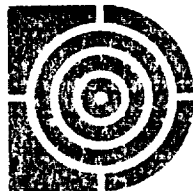
CSDL-R-1582

EQUATIONS OF MOTION FOR A FLEXIBLE
SPACECRAFT -- LUMPED PARAMETER IDEALIZATION

by
Joel Storch
Stephen Gates

September 1982

PROPERTY OF THE
TECHNICAL INFORMATION CENTER
CHARLES STARK DRAPER LABORATORY, INC.



The Charles Stark Draper Laboratory, Inc.

Cambridge, Massachusetts 02139

CNDR-R-1582 (27) DATE 10-1-82
FLEXIBLE SPACECRAFT-LUMPED PARAMETER
IDEALIZATION (Draper (Charles Storch) Lab.)
7/80 USCL 22

01-1111

Unclass

03/18 0033443

R-1582

PUBLICATION AUTHORIZATION FORM
For External Distribution and Formal Program Documents

TCO Work Order No.

Document No. assigned

8209H217

R-1582

DESCRIPTION — to be filled in by author

1. Title of document: <u>Equations of Motion for a Flexible Spacecraft - Lumped Parameter Idealizations</u>		2. Author(s): <u>Joel Storch</u> <u>Stephen Gates</u>	
3. Work generated under Lab account No. <u>92521</u>	4. Classification Unclassified <input checked="" type="checkbox"/> Confidential <input type="checkbox"/> Secret <input type="checkbox"/> S/RD <input type="checkbox"/>	5. Date(s) due: Draft copy _____ Final copies _____	
6. DOCUMENT TYPE <input checked="" type="checkbox"/> Technical Report (interim/final) R <input type="checkbox"/> Specification S <input type="checkbox"/> Technical Memorandum C <input type="checkbox"/> Thesis T <input type="checkbox"/> Technical Paper, slides, etc. P Remarks _____ To be prepared by: Publications (65E) <input checked="" type="checkbox"/> Other _____	7. DISTRIBUTION <input type="checkbox"/> List attached <input type="checkbox"/> List included in document Program _____ Internal _____ CSDL _____ Author(s) <u>2</u> TIC/Library _____ External: Gov't Agencies/Contractors _____ Other external <u>3</u> TOTAL <u>5</u> Remarks <u>Security Review</u> <u>3 Copies to Security</u>		8. PAPERS & PRESENTATIONS—public dissemination <input type="checkbox"/> Abstract, for submission/acceptance or oral presentation. — (resulting paper requires a separate authorization form) <input type="checkbox"/> Manuscript, for external publishing <input type="checkbox"/> Slides, vugraphs, etc. not included in a paper <input type="checkbox"/> Copies to be released at seminar/symposium <input type="checkbox"/> Oral presentation only To be published by _____ Society _____ Subject of meeting _____ Meeting date(s) _____ Location _____
9. Will any invention be disclosed that has not been submitted to CSDL Patent Committee? YES <input type="checkbox"/> NO <input checked="" type="checkbox"/>			
10. Is sponsor review required for technical content? YES <input checked="" type="checkbox"/> NO <input type="checkbox"/> Reviewing Agency _____		11. Charge this effort to Lab account No. <u>92521</u> Signed <u>Joel Storch</u> <u>9/82</u> Senior Author date	

AUTHORIZATION TO PREPARE — document described above

Signatures R. Storch 9/17/82 and Joseph E. Turnbull 9/27/82 if nonproject funded
Author's Supervisor date Program Manager date Chief Scientist date

APPROVAL TO PUBLISH — camera-ready copy/manuscript

TCO/Publications: CSDL Styleguide and/or sponsor specs met? YES <input checked="" type="checkbox"/> NO <input type="checkbox"/> <u>K. W. Storch</u> <u>9/27/82</u> Reviewing Editor date		Security Office: released as is <input type="checkbox"/> review required <input checked="" type="checkbox"/> <u>J. P. O'Neill</u> <u>9/27/82</u> Security Officer date	
camera-ready copy <u>Joel Storch</u> <u>9-27-82</u> Senior Author date	overall approval <u>J. P. O'Neill</u> <u>9-27-82</u> Department Head — Sponsor interface date		
technical approval <u>Joseph E. Turnbull</u> <u>9/27/82</u> Program Manager or Technical Supervisor date	IF thesis or IR&D Department Head — Education IR&D date		
overall approval <u>J. P. O'Neill</u> <u>9-27-82</u> Author's Department Head date	IF nonproject funded Chief Scientist date		

Remarks _____

DOCUMENT CONTROL:

received STM copies, from negs signed J. W. Storch 9/30/82
date

Distribution: Program _____ copies, on _____ Internal _____ copies, on _____ External _____ copies, on _____

TECHNICAL REPORT STANDARD TITLE PAGE

1. Report No.	2. Government Accession No.	3. Recipient's Catalog No.	
4. Title and Subtitle Equations of Motion for a Flexible Spacecraft — Lumped Parameter Idealization		5. Report Date September 1982	
		6. Performing Organization Code	
7. Author(s) Joel Storch, Stephen Gates		8. Performing Organization Report No. CSDL-R-1582	
9. Performing Organization Name and Address The Charles Stark Draper Laboratory, Inc. 555 Technology Square Cambridge, Massachusetts 02139		10. Work Unit No.	
		11. Contract or Grant No. NAS-9 16023	
12. Sponsoring Agency Name and Address National Aeronautics and Space Administration Johnson Space Center Houston, Texas 77058		13. Type of Report and Period Covered Technical Report	
		14. Sponsoring Agency Code	
15. Supplementary Notes			
16. Abstract The equations of motion for a flexible vehicle capable of arbitrary translational and rotational motions in inertial space accompanied by small elastic deformations are derived in an unabridged form. The vehicle is idealized as consisting of a single rigid body with an ensemble of mass particles interconnected by massless elastic structure. The internal elastic restoring forces are quantified in terms of a stiffness matrix. A transformation and truncation of elastic degrees of freedom is made in the interest of numerical integration efficiency. Deformation dependent terms are partitioned into a hierarchy of significance. The final set of motion equations are brought to a fully assembled first order form suitable for direct digital implementation. A FORTRAN program implementing the equations is given and its salient features described.			
17. Key Words Suggested by Author Spacecraft Dynamics Flexible Body Mechanics Lumped Parameter Modal Coordinates Mechanical Vibrations		18. Distribution Statement	
19. Security Classif. (of this report) UNCLASSIFIED	20. Security Classif. (of this page) UNCLASSIFIED	21. No. of Pages 74	22. Price

The final chapter of this report pertains to a FORTRAN computer program which implements and numerically integrates the complete set of equations. The salient features of the program, its subroutines, and the input and output data are described. An annotated flowchart along with a full listing of the code is provided.

It is noteworthy that while the idealization and methodology applied in this report are essentially those of Likins,⁽⁵⁾ the equations formulated herein are unique from those developed in Reference 5, and indeed the distinction is fundamental. It was the express desire to avoid the kinematic restrictions required there to effect a coordinate transformation on the elastic deflections which motivated this approach.

From an applications standpoint, the basic discretization of the vehicle of interest is performed in the manner of lumped mass structural dynamics modeling. The required stiffness matrix which quantifies the internal elastic restoring forces can in general be obtained from pre-processed linear structural finite element analysis programs (e.g., NASTRAN). Because of the mass particle idealization of the elastic domain, only translational displacements are defined at those points, hence any finite element model used to provide stiffness matrix information must be purged of any rotational degrees of freedom that may exist. This requirement is easily satisfied through the application of the static condensation procedure. Thus the analyst is afforded these familiar and versatile structural modeling techniques augmented by the arbitrary motion capability.

The motion equations formulated here are complete and unabridged for a single unconstrained flexible vehicle. However, they could, in a straightforward fashion, be coupled to the dynamics equations for other independent bodies to form an articulated system. This can be done through the identification and elimination of interbody constraint forces/torques and redundant kinematic variables. Indeed, it is for just such an application that these equations are intended. Specifically they are to represent a generic flexible payload to be terminally attached to the

Space Shuttle Orbiter remote manipulator system, which is an articulated chain of rigid and flexible bodies. For this case the model's rigid-body is taken to be the payload grapple fixture with all outboard structure represented by the particle assemblage.

CHAPTER 2

PRELIMINARIES

2.1 Vehicle Idealization

The system being analyzed (see Figure 1) consists of a single rigid body and an attached flexible appendage. The appendage is idealized as a system of particles connected by massless elastic structure. There is no articulation between the appendage and rigid base, i.e., the appendage is "cantilevered" to the rigid body. At an arbitrary point, O_g , of the rigid body we locate the origin of the body fixed frame which rotates as the body rotates in inertial space. The vector \vec{R} serves to determine the position of O_g relative to the inertially fixed point O . The particle masses m_i ($i = 1, 2, \dots, n$) are located via the position vectors \vec{r}_i relative to O_g in the undeformed state. The elastic displacement of m_i is \vec{q}_i , measured in the body frame.

Many space vehicles or parts of spacecraft can be approximated in this manner. A specific example is the Shuttle Remote Manipulation System in which the "appendage" corresponds to a flexible payload and the "rigid base" to the grapple fixture (this component being attached to the orbiter through the links of the manipulator arm).

2.2 External and Internal Forces

With the ultimate goal in mind of applying the present analysis to more complicated situations, we wish to accommodate all forces and torques which will arise when the system in Figure 1 is attached to other spacecraft components. Hence, at the point O_g , let there be a force-torque

pair: $\vec{f}^o(t)$, $\vec{\tau}^o(t)$. In the domain of the elastic appendage we have an external force $\vec{f}^i(t)$ acting upon the point mass m_i .

As an example, in the case of the Shuttle Remote Manipulator Arm, \vec{f}^o and $\vec{\tau}^o$ would represent the force and torque exerted by the end-effector on the grapple fixture.

We assume that we are given a stiffness matrix reflecting the mutual elastic forces between the mass points of the appendage. Assemble the elastic displacements as

$$\underline{q} = (q_x^1, q_y^1, q_z^1, q_x^2, q_y^2, q_z^2, \dots, q_x^n, q_y^n, q_z^n)^T$$

(x,y,z) refer to Cartesian components along the axes of the body fixed frame.

If $[K]$ is the stiffness matrix, then $-[K]\underline{q}$ is the vector of elastic forces exerted on the point masses. Assume that $[K]$ is partitioned such that the vector of elastic forces are ordered exactly as the elements in \underline{q} . Note that in generating $[K]$ the appendage is cantilevered to the rigid base.

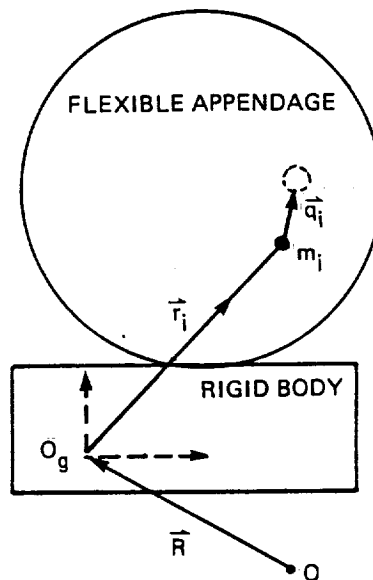


Figure 1. Idealized vehicle.

CHAPTER 3

EQUATIONS OF MOTION

3.1 Particle Translational Equations

\vec{v}^i , the inertial velocity of the i^{th} particle, is given by

$$\vec{v}^i = \frac{d}{dt} (\vec{R} + \vec{r}^i + \vec{q}^i)$$

Let $\begin{pmatrix} u \\ v \\ w \end{pmatrix}$ and $\underline{\omega}$ represent the (absolute) velocity and angular velocity of the body frame resolved in body axes, we then have

$$\underline{v}^i = \begin{pmatrix} u \\ v \\ w \end{pmatrix} + \underline{\omega} \times (\underline{r}^i + \underline{q}^i) + \dot{\underline{q}}^i$$

Differentiating this expression we arrive at the particle acceleration

$$\begin{aligned} \frac{d}{dt} \underline{v}^i &= \begin{pmatrix} \dot{u} \\ \dot{v} \\ \dot{w} \end{pmatrix} - (\underline{r}^i + \underline{q}^i) \times \dot{\underline{\omega}} + \ddot{\underline{q}}^i \\ &\quad + \underline{\omega} \times \left[\begin{pmatrix} u \\ v \\ w \end{pmatrix} + 2\dot{\underline{q}}^i \right] + \underline{\omega} \times [\underline{\omega} \times (\underline{r}^i + \underline{q}^i)] \end{aligned}$$

Expanding the cross product in the last term and using the matrix-vector form for the cross product $\underline{a} \times \underline{b} \equiv [\underline{a}]^{\sim} \underline{b}$ where $[\underline{a}]^{\sim} = \begin{pmatrix} 0 & -a_3 & a_2 \\ a_3 & 0 & -a_1 \\ -a_2 & a_1 & 0 \end{pmatrix}$ the particle acceleration can be written as

$$\begin{aligned} \frac{d}{dt} \underline{v}^i &= \begin{pmatrix} \dot{u} \\ \dot{v} \\ \dot{w} \end{pmatrix} - ([\underline{r}^i]^{\sim} + [\underline{q}^i]^{\sim}) \underline{\dot{\omega}} + \underline{\ddot{q}}^i + [\underline{\omega}]^{\sim} \begin{pmatrix} u \\ v \\ w \end{pmatrix} + 2[\underline{\omega}]^{\sim} \underline{\dot{q}}^i \\ &+ (\underline{\omega}^T \underline{r}^i) \underline{\omega} - ||\underline{\omega}||^2 \underline{r}^i + (\underline{\omega}^T \underline{q}^i) \underline{\omega} - ||\underline{\omega}||^2 \underline{q}^i \end{aligned} \quad (3-1)$$

If \underline{f}^i is the external force on the i^{th} particle and \underline{f}_e^i the elastic force exerted on the i^{th} particle by the rest of the assemblage (both resolved along body axes), the translational equation is

$$\begin{aligned} \underline{f}^i + \underline{f}_e^i &= m_i \frac{d}{dt} \underline{v}^i \\ (i &= 1, 2, \dots, n) \end{aligned}$$

Partitioning the stiffness matrix into (3x3) arrays

$$[\underline{K}] = \begin{pmatrix} [\underline{K}_{11}] & [\underline{K}_{12}] \dots [\underline{K}_{1n}] \\ [\underline{K}_{21}] & [\underline{K}_{22}] \dots [\underline{K}_{2n}] \\ \vdots & \vdots \quad \quad \quad \vdots \\ [\underline{K}_{n1}] & [\underline{K}_{n2}] \dots [\underline{K}_{nn}] \end{pmatrix}$$

$$\underline{f}_e^i = - \sum_{j=1}^n [\underline{K}_{ij}] \underline{q}^j$$

Employing Eq. (3-1) for the particle acceleration, the translation equations for the appendage particles may be assembled as

$$\begin{bmatrix} m^1 \\ m^2 \\ \vdots \\ m^n \end{bmatrix} \begin{pmatrix} \dot{u} \\ \dot{v} \\ \dot{w} \end{pmatrix} - \begin{bmatrix} m_1(\dot{\underline{r}}^1 + \dot{\underline{q}}^1) \\ m_2(\dot{\underline{r}}^2 + \dot{\underline{q}}^2) \\ \vdots \\ m_n(\dot{\underline{r}}^n + \dot{\underline{q}}^n) \end{bmatrix} \underline{\underline{\varepsilon}} + \begin{bmatrix} m^1 & 0 & 0 & \dots & 0 \\ 0 & m^2 & 0 & \dots & 0 \\ 0 & 0 & m^3 & \dots & \cdot \\ \cdot & \cdot & \cdot & \ddots & \cdot \\ 0 & 0 & \cdot & \cdot & m^n \end{bmatrix} \underline{\underline{q}} = \begin{bmatrix} \underline{r}^1 \\ \underline{r}^2 \\ \vdots \\ \underline{r}^n \end{bmatrix} + \underline{\underline{u}}_V \quad (3-2)$$

where we have introduced the symbol $m^i = \begin{bmatrix} m_i & 0 & 0 \\ 0 & m_i & 0 \\ 0 & 0 & m_i \end{bmatrix}$ ($i = 1, 2, \dots, n$)

and the nonlinear kinematic term $\underline{\underline{u}}_V$ is given by

$$\begin{aligned}
\underline{\underline{u}}_V = & - \begin{bmatrix} m_1 \underline{\underline{\omega}} \\ m_2 \underline{\underline{\omega}} \\ \vdots \\ m_n \underline{\underline{\omega}} \end{bmatrix} \begin{pmatrix} u \\ v \\ w \end{pmatrix} - 2 \begin{bmatrix} m_1 \underline{\underline{\omega}} \cdot \dot{\underline{q}}^1 \\ m_2 \underline{\underline{\omega}} \cdot \dot{\underline{q}}^2 \\ \vdots \\ m_n \underline{\underline{\omega}} \cdot \dot{\underline{q}}^n \end{bmatrix} - \begin{bmatrix} m_1 \underline{\underline{\omega}} \cdot (\underline{r}^1 + \underline{q}^1) \underline{\underline{\omega}} \\ m_2 \underline{\underline{\omega}} \cdot (\underline{r}^2 + \underline{q}^2) \underline{\underline{\omega}} \\ \vdots \\ m_n \underline{\underline{\omega}} \cdot (\underline{r}^n + \underline{q}^n) \underline{\underline{\omega}} \end{bmatrix} \\
& + ||\underline{\underline{\omega}}||^2 \begin{bmatrix} m_1(\underline{r}^1 + \underline{q}^1) \\ m_2(\underline{r}^2 + \underline{q}^2) \\ \vdots \\ m_n(\underline{r}^n + \underline{q}^n) \end{bmatrix} \quad (3-3)
\end{aligned}$$

Equation (3-2) constitutes a set of $3n$ scalar differential equations.

3.2 Vehicle Translational Equations

For the composite system (rigid body and appendage) the sum of the external forces equals the total mass times the acceleration of the mass center.

If m_b is the mass of the rigid body and \underline{s} is the vector from O_g to the mass center of the rigid body (expressed in the body frame)

$$\underline{m}\underline{c} = \sum_{i=1}^n m_i (\underline{r}_i^i + \underline{q}_i^i) + m_b \underline{s} \quad (3-4)$$

$m = m_b + \sum_{i=1}^n m_i$ is the total mass.

$\underline{c}(t)$ is the vector position of the instantaneous mass center relative to O_g .

The acceleration of the mass center is: $\frac{d^2}{dt^2} (\vec{R} + \vec{c})$

$$\frac{d^2 \vec{R}}{dt^2} = \begin{pmatrix} \dot{u} \\ \dot{v} \\ \dot{w} \end{pmatrix} + \underline{\omega} \times \begin{pmatrix} u \\ v \\ w \end{pmatrix} \quad (\text{in body frame})$$

$$m \frac{d^2 \underline{c}}{dt^2} = \sum_{i=1}^n m_i \ddot{\underline{q}}_i^i - \underline{m}\underline{c} \times \dot{\underline{\omega}} + 2\underline{\omega} \times \sum_{i=1}^n m_i \dot{\underline{q}}_i^i + \underline{\omega} \times (\underline{\omega} \times \underline{m}\underline{c})$$

Expressing this last term as: $(\underline{\omega} \cdot \underline{m}\underline{c})\underline{\omega} - ||\underline{\omega}||^2 \underline{m}\underline{c}$ the vehicle translational equation assumes the form

$$\begin{bmatrix} m & 0 & 0 \\ 0 & m & 0 \\ 0 & 0 & m \end{bmatrix} \begin{pmatrix} \dot{u} \\ \dot{v} \\ \dot{w} \end{pmatrix} - [\underline{m}\underline{c}] \dot{\underline{\omega}} + [m^1 \quad m^2 \quad \dots \quad m^n] \ddot{\underline{q}} = \sum_{i=0}^n \underline{f}_i^i + \underline{u}_t \quad (3-5)$$

The nonlinear term \underline{u}_t is given by

$$\underline{u}_t = -\underline{m}\underline{\omega} \times \begin{pmatrix} u \\ v \\ w \end{pmatrix} - 2\underline{\omega} \times \sum_{i=1}^n m_i \dot{\underline{q}}^i - (\underline{\omega} \cdot \underline{m}\underline{c})\underline{\omega} + \|\underline{\omega}\|^2 \underline{m}\underline{c} \quad (3-6)$$

3.3 Vehicle Rotational Equations

For the composite system (rigid body and appendage) the sum of the external torques taken about the mass center equals the time rate of change of the angular momentum taken about the mass center.

Let $[I_b]$ be the inertia matrix of the rigid body with respect to a coordinate system located at the mass center of the rigid body and parallel to the body fixed axes system at O_g .

$$[I_b] = \iiint [\underline{\lambda} \cdot \underline{\lambda} E - \underline{\lambda} \underline{\lambda}^T] dm$$

$\underline{\lambda}$ is the position vector of a mass element dm in the rigid body relative to the rigid body mass center and the integration is performed over the region occupied by the rigid base. $[E]$ denotes the unit matrix.

The system angular momentum can be split into two parts:

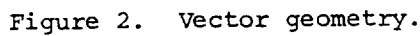
\underline{H}_b - angular momentum of rigid base

$\sum_{i=1}^n \underline{H}_i$ - angular momentum of appendage particles

Let $\underline{\ell}^i$ and $\underline{\ell}$ denote the position vectors from the system mass center to m_i and a generic mass element in the rigid body respectively.

$$\underline{H}_b = \iiint \underline{\ell} \times \frac{d}{dt} \underline{\ell} dm$$

$$\underline{H}_i = \underline{\ell}^i \times m_i \frac{d}{dt} \underline{\ell}^i$$



Inserting this expression for \underline{l} into the integral definition of \underline{H}_b and recalling the definition of $[\underline{I}_b]$ and the fact that $\iiint \underline{\lambda} \, d\mathbf{m} = \underline{0}$ we arrive at the following expression for \underline{H}_b

Thus

12

Turning to the angular momentum of the i^{th} particle

$$\frac{d}{dt} \underline{H}^i = \underline{\ell}^i \times m_i \frac{d^2}{dt^2} \underline{\ell}^i$$

Now

$$\begin{aligned} \frac{d^2}{dt^2} \underline{\ell}^i &= -(\underline{r}^i + \underline{q}^i) \times \dot{\underline{\omega}} + \ddot{\underline{q}}^i + 2\underline{\omega} \times \dot{\underline{q}}^i + [\underline{\omega} \cdot (\underline{r}^i + \underline{q}^i)] \underline{\omega} \\ &\quad - ||\underline{\omega}||^2 (\underline{r}^i + \underline{q}^i) - \frac{d^2}{dt^2} \underline{c} \end{aligned}$$

Combining the above expressions for $\frac{d}{dt} \underline{H}_b$ and $\frac{d}{dt} \underline{H}^i$ (with the substitution for $\frac{d^2}{dt^2} \underline{\ell}^i$) the terms involving $\frac{d^2}{dt^2} \underline{c}$ conveniently cancel leaving the following result for the time derivative of the angular momentum

$$\begin{aligned} \frac{d}{dt} \underline{H} &= ([I_b] - m_b (\underline{s} - \underline{c}) \sim \underline{\tilde{s}}) \dot{\underline{\omega}} + \underline{\omega} \times [I_b] \underline{\omega} + \sum_{i=1}^n m_i [\underline{\ell}^i] \sim \ddot{\underline{q}}^i \\ &\quad + m_b (\underline{s} - \underline{c}) \times [\underline{\omega} \times (\underline{\omega} \times \underline{s})] - \sum_{i=1}^n m_i [\underline{\ell}^i] \sim (\underline{r}^i + \underline{q}^i) \sim \dot{\underline{\omega}} \\ &\quad + 2 \sum_{i=1}^n m_i [\underline{\ell}^i] \sim \underline{\omega} \dot{\underline{q}}^i + \sum_{i=1}^n [\underline{\omega} \cdot (\underline{r}^i + \underline{q}^i)] m_i \underline{\ell}^i \times \underline{\omega} \\ &\quad - ||\underline{\omega}||^2 \sum_{i=1}^n m_i \underline{\ell}^i \times (\underline{r}^i + \underline{q}^i) \end{aligned} \quad (3-7)$$

The system rotational motion equation is

$$\begin{aligned}\frac{d}{dt} \underline{H} &= \underline{r}^0 - \underline{c} \times \underline{f}^0 + \sum_{i=1}^n (\underline{r}^i + \underline{q}^i - \underline{c}) \times \underline{f}^i \\ &= \underline{r}^0 - \underline{c} \times \sum_{j=0}^n \underline{f}^j + \sum_{i=1}^n (\underline{r}^i + \underline{q}^i) \times \underline{f}^i\end{aligned}$$

If we were to insert Eq. (3-7) into this last equation we would have a valid equation for system rotation. Note, however, that this equation would not depend upon $(\dot{u}, \dot{v}, \dot{w})$ explicitly. Since the translational equations (3-5) depend upon $\dot{\underline{\omega}}$ explicitly, and we wish to have a final set of equations with a symmetric coefficient matrix of the generalized accelerations, we force the coupling between the rotational equations and the acceleration vector $(\dot{u} \ \dot{v} \ \dot{w})$ by the following device.

Take Eq. (3-5) and (3-6) and solve for $\sum_{j=0}^n \underline{f}^j$. The system rotational motion equation then becomes

$$\begin{aligned}\frac{d}{dt} \underline{H} &= \underline{r}^0 + \sum_{i=1}^n (\underline{r}^i + \underline{q}^i) \times \underline{f}^i - [\underline{mc}]^{\sim} \begin{pmatrix} \dot{u} \\ \dot{v} \\ \dot{w} \end{pmatrix} + [\underline{c}]^{\sim} [\underline{mc}]^{\sim} \dot{\underline{\omega}} \\ &\quad - \underline{c} \times \sum_{i=1}^n m_i \ddot{\underline{q}}^i - [\underline{mc}]^{\sim} \tilde{\underline{\omega}} \begin{pmatrix} u \\ v \\ w \end{pmatrix} - 2[\underline{c}]^{\sim} \tilde{\underline{\omega}} \sum_{i=1}^n m_i \dot{\underline{q}}^i \\ &\quad - (\underline{\omega} \cdot \underline{mc}) \underline{c} \times \underline{\omega}\end{aligned}\tag{3-8}$$

Combining Eq. (3-7) and (3-8), we arrive at the final desired form for the vehicle rotational equation

$$\begin{aligned}
[m\bar{c}]^{\sim} \begin{pmatrix} \dot{u} \\ \dot{v} \\ \dot{w} \end{pmatrix} + [I(t)] \dot{\bar{\omega}} + \left[m_1(\underline{r}^1 + \underline{q}^1)^{\sim} | m_2(\underline{r}^2 + \underline{q}^2)^{\sim} | \dots | m_n(\underline{r}^n + \underline{q}^n)^{\sim} \right] \ddot{\underline{q}} \\
= \underline{r}^0 + \sum_{i=1}^n (\underline{r}^i + \underline{q}^i) \times \underline{f}^i + \underline{u}_r
\end{aligned} \tag{3-9}$$

Here

$$[I(t)] = [I_b] - m_b(\underline{s} - \underline{c})^{\sim} \underline{\tilde{s}} - \sum_{i=1}^n m_i [\underline{\ell}^i]^{\sim} (\underline{r}^i + \underline{q}^i)^{\sim} - [\tilde{c}] [m\bar{c}]^{\sim}$$

which can be simplified to

$$[I(t)] = [I_b] - m_b[\underline{\tilde{s}}]^2 - \sum_{i=1}^n m_i (\underline{\tilde{r}}^i + \underline{\tilde{q}}^i)^2 \tag{3-10}$$

In this form we recognize $[I(t)]$ as the inertia matrix of the vehicle about point O_g .

The nonlinear rotation term \underline{u}_r is given by

$$\begin{aligned}
\underline{u}_r &= -[m\bar{c}]^{\sim} \underline{\tilde{\omega}} \begin{pmatrix} u \\ v \\ w \end{pmatrix} - 2[\underline{c}]^{\sim} [\underline{\omega}]^{\sim} \sum_{i=1}^n m_i \dot{\underline{q}}^i - (\underline{\omega} \cdot m\bar{c}) [\underline{c}]^{\sim} \underline{\omega} \\
&- [\underline{\omega}]^{\sim} [I_b] \underline{\omega} - m_b(\underline{s} - \underline{c}) \times [\underline{\omega} \times (\underline{\omega} \times \underline{s})] - 2 \sum_{i=1}^n m_i [\underline{\ell}^i]^{\sim} [\underline{\omega}]^{\sim} \dot{\underline{q}}^i \\
&- \sum_{i=1}^n [\underline{\omega} \cdot (\underline{r}^i + \underline{q}^i)] m_i \underline{\ell}^i \times \underline{\omega} + ||\underline{\omega}||^2 \sum_{i=1}^n m_i \underline{\ell}^i \times (\underline{r}^i + \underline{q}^i)
\end{aligned} \tag{3-11}$$

CHAPTER 4

EXPANSION AND PARTITIONING OF TERMS

4.1 Deformation Dependent Coefficients

In the equations developed thus far, specifically Eq. (3-2), (3-5), and (3-9), we have isolated the accelerations on the left hand sides of the respective equations. The acceleration coefficients are time dependent through the elastic deformations. It is quite desirable from an applications viewpoint to rank the constituents in these coefficients in accordance with their relative magnitude. Thus, in a computer simulation, one can choose to omit certain terms and speed up execution with a minimal impact on computed results.

We will rank terms amongst three categories:

- (1) Terms independent of \underline{q} .
- (2) Terms first order in \underline{q} .
- (3) Terms second order in \underline{q} .

The majority of coefficients are directly identifiable in this hierarchy. We have two coefficients which require additional attention: \underline{m}_c and $[I(t)]$.

From Eq. (3-4), $\underline{m}_c = \underline{m}_b \underline{s} + \sum_{i=1}^n \underline{m}_i \underline{r}^i + \sum_{i=1}^n \underline{m}_i \underline{q}^i(t)$. The first two terms are of category 1 and the sum will be denoted by \underline{m}_{c0} . The time-dependent term will be denoted by $\underline{m}_{c1}(t)$.

The matrix $[I(t)]$ is given by Eq. (3-10) and can be written as

$$[I(t)] = [I_1] + [I_2(t)] + [I_3(t)]$$

The three matrices $[I_1]$, $[I_2(t)]$, and $[I_3(t)]$ are of category 1, 2, and 3 respectively and are given by

$$[I_1] = [I_b] - m_b [\underline{\tilde{s}}]^2 - \sum_{i=1}^n m_i [\underline{\tilde{r}}^i]^2$$

$$[I_2(t)] = - \sum_{i=1}^n m_i ([\underline{\tilde{r}}^i] [\underline{\tilde{q}}^i] + [\underline{\tilde{q}}^i] [\underline{\tilde{r}}^i])$$

$$[I_3(t)] = - \sum_{i=1}^n m_i [\underline{\tilde{q}}^i]^2$$

4.2 Nonlinear Kinematic Terms

In this section, we concentrate upon the three nonlinear terms: \underline{u}_t , \underline{u}_v , and \underline{u}_r appearing on the right hand sides of the motion equations. Following a procedure similar to that of the previous section, the nonlinear terms are partitioned amongst three categories:

- (1) Nonlinear terms independent of \underline{q} , $\dot{\underline{q}}$
- (2) Nonlinear terms first order in \underline{q} , $\dot{\underline{q}}$
- (3) Nonlinear terms second order in \underline{q} , $\dot{\underline{q}}$

Accordingly, from Eq. (3-6), $\underline{u}_t = \underline{u}_t^{(1)} + \underline{u}_t^{(2)}$ with

$$\underline{u}_t^{(1)} = -m[\underline{\omega}] \sim \begin{pmatrix} u \\ v \\ w \end{pmatrix} - (\underline{\omega} \cdot m\underline{c}_0) \underline{\omega} + ||\underline{\omega}||^2 m\underline{c}_0 \quad (4-1)$$

$$\underline{u}_t^{(2)} = -2[\underline{\omega}] \sim \sum_{i=1}^n m_i \dot{\underline{q}}^i - (\underline{\omega} \cdot m\underline{c}_1) \underline{\omega} + ||\underline{\omega}||^2 m\underline{c}_1 \quad (4-2)$$

In a similar manner from Eq. (3-3)

$$\underline{u}_v = \underline{u}_v^{(1)} + \underline{u}_v^{(2)}$$

$$\underline{u}_v^{(1)} = - \begin{bmatrix} m_1 \underline{\omega} \\ m_2 \underline{\omega} \\ \vdots \\ m_n \underline{\omega} \end{bmatrix} \begin{pmatrix} u \\ v \\ w \end{pmatrix} - \begin{bmatrix} m_1 (\underline{\omega} \cdot \underline{r}^1) \underline{\omega} \\ m_2 (\underline{\omega} \cdot \underline{r}^2) \underline{\omega} \\ \vdots \\ m_n (\underline{\omega} \cdot \underline{r}^n) \underline{\omega} \end{bmatrix} + ||\underline{\omega}||^2 \begin{bmatrix} m_1 \underline{r}^1 \\ m_2 \underline{r}^2 \\ \vdots \\ m_n \underline{r}^n \end{bmatrix} \quad (4-3)$$

$$\underline{u}_v^{(2)} = -2 \begin{bmatrix} m_1 \underline{\omega} \cdot \dot{\underline{q}}^1 \\ m_2 \underline{\omega} \cdot \dot{\underline{q}}^2 \\ \vdots \\ m_n \underline{\omega} \cdot \dot{\underline{q}}^n \end{bmatrix} - \begin{bmatrix} m_1 (\underline{\omega} \cdot \underline{q}^1) \underline{\omega} \\ m_2 (\underline{\omega} \cdot \underline{q}^2) \underline{\omega} \\ \vdots \\ m_n (\underline{\omega} \cdot \underline{q}^n) \underline{\omega} \end{bmatrix} + ||\underline{\omega}||^2 \begin{bmatrix} m_1 \underline{q}^1 \\ m_2 \underline{q}^2 \\ \vdots \\ m_n \underline{q}^n \end{bmatrix} \quad (4-4)$$

Expansion of \underline{u}_r

The two terms in \underline{u}_r (Eq. (3-11)) depending upon $\dot{\underline{q}}$ can be combined as

$$\begin{aligned} -2 \sum_{i=1}^n m_i ([\underline{c}]^{\sim} + [\underline{l}^i]^{\sim}) [\underline{\omega}]^{\sim} \dot{\underline{q}}^i &= -2 \sum_{i=1}^n m_i (\underline{r}^i + \underline{q}^i)^{\sim} [\underline{\omega}]^{\sim} \dot{\underline{q}}^i \\ &= -2 \sum_{i=1}^n m_i [\underline{r}^i]^{\sim} [\underline{\omega}]^{\sim} \dot{\underline{q}}^i - 2 \sum_{i=1}^n m_i [\underline{q}^i]^{\sim} [\underline{\omega}]^{\sim} \dot{\underline{q}}^i \end{aligned}$$

The third term in \underline{u}_r can be expressed as

$$\begin{aligned} (\underline{\omega} \cdot \underline{m}\underline{c}) [\underline{c}] \sim \underline{\omega} &= (\underline{\omega} \cdot \underline{m}\underline{c}_0) [\underline{c}_0] \sim \underline{\omega} + (\underline{\omega} \cdot \underline{m}\underline{c}_0) [\underline{c}_1] \sim \underline{\omega} + (\underline{\omega} \cdot \underline{m}\underline{c}_1) [\underline{c}_0] \sim \underline{\omega} \\ &+ (\underline{\omega} \cdot \underline{m}\underline{c}_1) [\underline{c}_1] \sim \underline{\omega} \end{aligned}$$

For the last two terms in \underline{u}_r the following expansions are useful

$$\begin{aligned} [\underline{\omega} \cdot (\underline{r}^i + \underline{q}^i)] m_i \underline{\ell}^i \times \underline{\omega} &= (\underline{\omega} \cdot \underline{r}^i) m_i (\underline{r}^i - \underline{c}_0) \times \underline{\omega} \\ &+ (\underline{\omega} \cdot \underline{q}^i) m_i (\underline{r}^i - \underline{c}_0) \times \underline{\omega} \\ &+ (\underline{\omega} \cdot \underline{r}^i) m_i (\underline{q}^i - \underline{c}_1) \times \underline{\omega} \\ &+ (\underline{\omega} \cdot \underline{q}^i) m_i (\underline{q}^i - \underline{c}_1) \times \underline{\omega} \end{aligned}$$

$$\underline{\ell}^i \times (\underline{r}^i + \underline{q}^i) = \underline{r}^i \times \underline{c}_0 + \underline{r}^i \times \underline{c}_1 + \underline{q}^i \times \underline{c}_0 + \underline{q}^i \times \underline{c}_1$$

Collecting terms in \underline{u}_r independent of deformation

$$\begin{aligned} \underline{u}_r^{(1)} &= -[\underline{m}\underline{c}_0] \sim [\underline{\omega}] \sim \begin{pmatrix} u \\ v \\ w \end{pmatrix} - (\underline{\omega} \cdot \underline{m}\underline{c}_0) [\underline{c}_0] \sim \underline{\omega} - \underline{\omega} \times [\underline{I}_b] \underline{\omega} \\ &- m_b (\underline{s} - \underline{c}_0) \times [\underline{\omega} \times (\underline{\omega} \times \underline{s})] - \sum_{i=1}^n m_i (\underline{\omega} \cdot \underline{r}^i) (\underline{r}^i - \underline{c}_0) \times \underline{\omega} \\ &+ ||\underline{\omega}||^2 \sum_{i=1}^n m_i \underline{r}^i \times \underline{c}_0 \end{aligned}$$

The expression for $\underline{u}_r^{(1)}$ can be further simplified by use of the following identities

$$\sum_{i=1}^n m_i \underline{r}^i \times \underline{c}_0 = -m_b \underline{s} \times \underline{c}_0$$

$$\begin{aligned} \sum_{i=1}^n m_i (\underline{\omega} \cdot \underline{r}^i) (\underline{r}^i - \underline{c}_0) \times \underline{\omega} &= -[\underline{\omega}]^{\sim} \sum_{i=1}^n m_i \underline{r}^i \underline{r}^{iT} \underline{\omega} \\ &+ [\underline{\omega} \cdot (m_b \underline{s} - m \underline{c}_0)] \underline{c}_0 \times \underline{\omega} \end{aligned}$$

$$\begin{aligned} -m_b (\underline{s} - \underline{c}_0) \times [\underline{\omega} \times (\underline{\omega} \times \underline{s})] &= -(\underline{\omega} \cdot \underline{s}) m_b \underline{s} \times \underline{\omega} + (\underline{\omega} \cdot \underline{s}) m_b \underline{c}_0 \times \underline{\omega} \\ &- ||\underline{\omega}||^2 m_b \underline{c}_0 \times \underline{s} \end{aligned}$$

Incorporating these results we arrive at the final expression

$$\begin{aligned} \underline{u}_r^{(1)} &= -[m \underline{c}_0]^{\sim} [\underline{\omega}]^{\sim} \begin{pmatrix} u \\ v \\ w \end{pmatrix} + [\underline{\omega}]^{\sim} \sum_{i=1}^n m_i \underline{r}^i \underline{r}^{iT} \underline{\omega} - \underline{\omega} \times [I_b] \underline{\omega} \\ &- m_b (\underline{s} \cdot \underline{\omega}) \underline{s} \times \underline{\omega} \end{aligned} \quad (4-5)$$

Collecting first order deformation dependent terms in \underline{u}_r

$$\begin{aligned} \underline{u}_r^{(2)} &= -[m \underline{c}_1]^{\sim} [\underline{\omega}]^{\sim} \begin{pmatrix} u \\ v \\ w \end{pmatrix} - 2 \sum_{i=1}^n m_i [\underline{r}^i]^{\sim} [\underline{\omega}]^{\sim} \underline{q}^i - (\underline{\omega} \cdot m \underline{c}_0) [\underline{c}_1]^{\sim} \underline{\omega} \\ &- (\underline{\omega} \cdot m \underline{c}_1) [\underline{c}_0]^{\sim} \underline{\omega} + m_b \underline{c}_1 \times [\underline{\omega} \times (\underline{\omega} \times \underline{s})] \\ &- \sum_{i=1}^n [(\underline{\omega} \cdot \underline{q}^i) m_i (\underline{r}^i - \underline{c}_0) \times \underline{\omega} + (\underline{\omega} \cdot \underline{r}^i) m_i (\underline{q}^i - \underline{c}_1) \times \underline{\omega}] \\ &+ ||\underline{\omega}||^2 \sum_{i=1}^n m_i (\underline{r}^i \times \underline{c}_1 + \underline{q}^i \times \underline{c}_0) \end{aligned}$$

The expression for $\underline{u}_r^{(2)}$ can be further simplified by use of the following identities

$$\begin{aligned} \sum_{i=1}^n m_i (\underline{r}^i \times \underline{c}_1 + \underline{q}^i \times \underline{c}_0) &= m_b \underline{c}_1 \times \underline{s} \\ - \sum_{i=1}^n [(\underline{\omega} \cdot \underline{q}^i) m_i (\underline{r}^i - \underline{c}_0) \times \underline{\omega} + (\underline{\omega} \cdot \underline{r}^i) m_i (\underline{q}^i - \underline{c}_1) \times \underline{\omega}] &= \\ [\underline{\omega}]^{\sim} \sum_{i=1}^n m_i (\underline{r}^i \underline{q}^{iT} + \underline{q}^i \underline{r}^{iT}) \underline{\omega} + (\underline{\omega} \cdot m \underline{c}_1) \underline{c}_0 \times \underline{\omega} + [\underline{\omega} \cdot (m \underline{c}_0 - m_b \underline{s})] \underline{c}_1 \times \underline{\omega} \\ m_b \underline{c}_1 \times [\underline{\omega} \times (\underline{\omega} \times \underline{s})] &= m_b (\underline{\omega} \cdot \underline{s}) \underline{c}_1 \times \underline{\omega} - m_b ||\underline{\omega}||^2 \underline{c}_1 \times \underline{s} \\ \underline{u}_r^{(2)} &= -[m \underline{c}_1]^{\sim} [\underline{\omega}]^{\sim} \begin{pmatrix} u \\ v \\ w \end{pmatrix} - 2 \sum_{i=1}^n m_i [\underline{r}^i]^{\sim} [\underline{\omega}]^{\sim} \underline{q}^i \\ &+ [\underline{\omega}]^{\sim} \sum_{i=1}^n m_i (\underline{r}^i \underline{q}^{iT} + \underline{q}^i \underline{r}^{iT}) \underline{\omega} \end{aligned} \quad (4-6)$$

Collecting second order deformation dependent terms in \underline{u}_r and simplifying

$$\underline{u}_r^{(3)} = -2 \sum_{i=1}^n m_i [\underline{q}^i]^{\sim} [\underline{\omega}]^{\sim} \underline{q}^i + [\underline{\omega}]^{\sim} \sum_{i=1}^n m_i \underline{q}^i \underline{q}^{iT} \underline{\omega} \quad (4-7)$$

$\underline{u}_r = \underline{u}_r^{(1)} + \underline{u}_r^{(2)} + \underline{u}_r^{(3)}$ where the terms on the right hand side are given by Eq. (4-5) - (4-7).

CHAPTER 5

MODAL COORDINATE TRANSFORMATION

When the number of particles in the appendage idealization becomes large, high frequencies obtain which make numerical integration difficult. We will describe a truncated coordinate transformation to circumvent this difficulty. Note that the treatment to follow is somewhat heuristic and hence requires good engineering judgement and caution in its implementation.

Since the high frequencies arise from the appendage vibration, it is natural to start with its governing equation (3-2, 3-3). Consider the case where no external forces act, $\underline{\omega} = \underline{0}$ and $(\dot{\underline{u}}, \dot{\underline{v}}, \dot{\underline{w}}) = \underline{0}$. The "constrained" appendage equation then assumes the familiar form

$$[M]\ddot{\underline{q}} + [K]\underline{q} = \underline{0} \quad \text{where } [M] = \text{diag}(m^1, m^2, \dots, m^n)$$

The natural frequencies, ω_i , and corresponding mode shapes, \underline{v}^i , are determined from

$$([K] - \omega^2[M])\underline{v} = \underline{0} \quad (5-1)$$

For the vehicle we are treating here, the appendage is rigidly attached to the base body hence no rigid body modes are present in the above eigenvalue problem. Equivalently, $[K]$ is positive definite. Since $[K]$ and $[M]$ are symmetric and positive definite there exists $3n$ independent eigenvectors \underline{v}^i corresponding to positive eigenvalues ω_i^2 , even if there are multiple eigenvalues.

We assume that the eigenvectors are normalized such that $(\underline{v}^i, [M]\underline{v}^i) = 1$. It follows that $(\underline{v}^i, [K]\underline{v}^i) = \omega_i^2$. We can always create a mutually orthogonal set such that

$$(\underline{v}^i, [M]\underline{v}^j) = 0 = (\underline{v}^i, [K]\underline{v}^j) \quad (i \neq j)$$

In actual computation we can deal with a simpler eigenvalue problem than that presented by Eq. (5-1). Specifically we will transform Eq. (5-1) to an ordinary symmetric eigenvalue problem. Introduce the change of variables: $\underline{W} = [M]^{1/2}\underline{V}$. Since $[M]$ is diagonal, $[M]^{1/2}$ is a diagonal matrix whose elements are the square roots of the corresponding elements in $[M]$. The eigenvalue problem transforms into

$$[K][M]^{-1/2}\underline{W} = \omega^2[M]^{1/2}\underline{W} \quad \text{or} \quad [\mathcal{G}]\underline{W} = \omega^2\underline{W} \quad (5-2)$$

where $[\mathcal{G}]$ is the symmetric matrix: $[\mathcal{G}] = [M]^{-1/2}[K][M]^{-1/2}$

It is easily verified that if the eigenvectors \underline{W}^i ($i = 1, 2, \dots, 3n$) of Eq. (5-2) are orthogonal (which can always be done) then the corresponding eigenvectors of (5-1) satisfy all orthogonality and normality conditions specified above.

Order the eigenvalues such that $\omega_1^2 \leq \omega_2^2 \leq \dots \leq \omega_{3n}^2$ and let $[\Phi]$ be the $(3n \times t)$ matrix whose columns are the eigenvectors $\underline{v}_1, \underline{v}_2, \dots, \underline{v}_t$ ($t \leq 3n$). We now make the transformation

$$\underline{q} = [\Phi]\underline{\eta} \quad (5-3)$$

This is not a coordinate transformation in the strict sense, since $[\Phi]$ does not have an inverse when $t < 3n$. The appendage deformation is now characterized by t "modal coordinates" instead of the original $3n$ deformation coordinates. We formally make the substitution, Eq. (5-3), into the full set of motion equations.

Substituting into the appendage deformation equation (3-2), pre-multiplying by $[\Phi]^T$ and recalling the orthogonality and normality conditions we arrive at

$$[\Phi]^T \begin{bmatrix} m^1 \\ m^2 \\ \vdots \\ m^n \end{bmatrix} \begin{pmatrix} \dot{u} \\ \dot{v} \\ \dot{w} \end{pmatrix} - [\Phi]^T \begin{bmatrix} m_1(\underline{r}^1 + \underline{q}^1)^\sim \\ m_2(\underline{r}^2 + \underline{q}^2)^\sim \\ \vdots \\ m_n(\underline{r}^n + \underline{q}^n)^\sim \end{bmatrix} \underline{\dot{\omega}} + \underline{\ddot{\eta}} + \begin{bmatrix} \omega_1^2 & 0 & 0 & \dots & 0 \\ 0 & \omega_2^2 & & & 0 \\ 0 & & \ddots & & \\ \vdots & & & \ddots & \\ 0 & 0 & \dots & \dots & \omega_t^2 \end{bmatrix} \underline{\eta} =$$

$$[\Phi]^T \begin{bmatrix} \underline{f}^1 \\ \underline{f}^2 \\ \vdots \\ \underline{f}^n \end{bmatrix} + [\Phi]^T \underline{u}_v \quad (5-4)$$

The vehicle translational equations (3-5) become

$$\begin{bmatrix} m & 0 & 0 \\ 0 & m & 0 \\ 0 & 0 & m \end{bmatrix} \begin{pmatrix} \dot{u} \\ \dot{v} \\ \dot{w} \end{pmatrix} - [m\underline{c}]^\sim \underline{\dot{\omega}} + [m^1 \ m^2 \ \dots \ m^n] [\Phi] \underline{\ddot{\eta}} = \sum_{i=0}^n \underline{f}^i + \underline{u}_t$$

The vehicle rotational equations (3-9) become

$$[m\underline{c}]^\sim \begin{pmatrix} \dot{u} \\ \dot{v} \\ \dot{w} \end{pmatrix} + [I(t)] \underline{\dot{\omega}} + [m_1(\underline{r}^1 + \underline{q}^1)^\sim \mid m_2(\underline{r}^2 + \underline{q}^2)^\sim \mid \dots \mid m_n(\underline{r}^n + \underline{q}^n)^\sim] [\Phi] \underline{\ddot{\eta}} =$$

$$\underline{\tau}^0 + \sum_{i=1}^n (\underline{r}^i + \underline{q}^i) \times \underline{f}^i + \underline{u}_r$$

The assembled equations of motion in matrix form are presented in Figure 3.

$$\left[\begin{array}{c|c|c} \begin{bmatrix} m & 0 & 0 \\ 0 & m & 0 \\ 0 & 0 & m \end{bmatrix} & -[m\underline{c}]^{\sim} & [m^1 m^2 \dots m^n] [\Phi] \\ \hline [\underline{m\underline{c}}]^{\sim} & [l(t)] & [m_1(\underline{r}^1 + \underline{q}^1)^{\sim} \dots m_n(\underline{r}^n + \underline{q}^n)^{\sim}] [\Phi] \\ \hline [\Phi]^T \begin{bmatrix} m^1 \\ m^2 \\ \vdots \\ m^n \end{bmatrix} & -[\Phi]^T \begin{bmatrix} m_1(\underline{r}^1 + \underline{q}^1)^{\sim} \\ m_2(\underline{r}^2 + \underline{q}^2)^{\sim} \\ \vdots \\ m_n(\underline{r}^n + \underline{q}^n)^{\sim} \end{bmatrix} & \begin{bmatrix} 1 & 0 & 0 & \dots & 0 & 0 \\ 0 & 1 & 0 & \dots & 0 & 0 \\ 0 & 0 & 1 & \dots & 0 & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots & \vdots \\ 0 & 0 & 0 & \dots & 1 \end{bmatrix} \end{array} \right] \begin{bmatrix} \underline{u} \\ \underline{v} \\ \underline{w} \\ \underline{\xi} \\ \underline{\eta} \end{bmatrix}$$

$$= \left[\begin{array}{c} \sum_{i=0}^n \underline{f}^i \\ \hline \underline{x}^0 + \sum_{i=1}^n (\underline{r}^i + \underline{q}^i) \times \underline{f}^i \\ \hline [\Phi]^T \begin{pmatrix} \underline{f}^1 \\ \underline{f}^2 \\ \vdots \\ \underline{f}^n \end{pmatrix} \end{array} \right] - \left[\begin{array}{c} \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix} \\ \hline \begin{pmatrix} 0 \\ 0 \\ 0 \end{pmatrix} \\ \hline \begin{bmatrix} \omega_1^2 & 0 & 0 & \dots & 0 \\ 0 & \omega_2^2 & & & \\ 0 & & \ddots & & \\ \vdots & & & \ddots & \\ 0 & 0 & \dots & \dots & \omega_t^2 \end{bmatrix} \underline{\eta} \end{array} \right] + \left[\begin{array}{c} \underline{u}_t \\ \hline \underline{u}_r \\ \hline [\Phi]^T \underline{u}_v \end{array} \right]$$

Figure 3. Assembled equations of motion.

CHAPTER 6

SYSTEM KINETIC ENERGY

The kinetic energy of the vehicle is the sum of the translational and rotational kinetic energy of the rigid body and the kinetic energy of the particles comprising the appendage

$$T = \frac{1}{2} m_b v_b^2 + \frac{1}{2} \underline{\omega} \cdot [I_b] \underline{\omega} + \frac{1}{2} \sum_{i=1}^n m_i v_i^2$$

$\underline{v}_b = \begin{pmatrix} u \\ v \\ w \end{pmatrix} + \underline{\omega} \times \underline{s}$ is the velocity of the mass center of the base

$\underline{v}^i = \begin{pmatrix} u \\ v \\ w \end{pmatrix} + \underline{\omega} \times (\underline{r}^i + \underline{q}^i) + \dot{\underline{q}}^i$ is the velocity of i^{th} particle

Forming the inner products $(\underline{v}_b, \underline{v}_b); (\underline{v}^i, \underline{v}^i)$ and recalling Eq. (3-10) for $[I(t)]$ the kinetic energy can be written as

$$\begin{aligned} T = & \frac{1}{2} m(u^2 + v^2 + w^2) + \frac{1}{2} \underline{\omega}^T [I(t)] \underline{\omega} + \frac{1}{2} \sum_{i=1}^n \{\dot{\underline{q}}^i\}^T m_i \dot{\underline{q}}^i \\ & - \frac{1}{2} (uvw) [m\underline{c}]^T \underline{\omega} + \frac{1}{2} \underline{\omega}^T [m\underline{c}] \begin{pmatrix} u \\ v \\ w \end{pmatrix} + \frac{1}{2} (uvw) \sum_{i=1}^n m_i \dot{\underline{q}}^i \\ & + \frac{1}{2} \sum_{i=1}^n m_i \{\dot{\underline{q}}^i\}^T \begin{pmatrix} u \\ v \\ w \end{pmatrix} + \frac{1}{2} \underline{\omega}^T \sum_{i=1}^n m_i (\underline{r}^i + \underline{q}^i) \dot{\underline{q}}^i - \frac{1}{2} \sum_{i=1}^n m_i \{\dot{\underline{q}}^i\}^T (\underline{r}^i + \underline{q}^i) \underline{\omega} \end{aligned}$$

We now rewrite those terms in T which depend upon $\dot{\underline{q}}^i$ in terms of $\dot{\underline{n}}$

$$\sum_{i=1}^n \{\dot{\underline{q}}^i\}^T m_i \dot{\underline{q}}^i = \dot{\underline{q}}^T [M] \dot{\underline{q}} = \dot{\underline{n}}^T [\Phi]^T [M] [\Phi] \dot{\underline{n}} = \dot{\underline{n}}^T [E] \dot{\underline{n}}$$

$[E]$ is the $(t \times t)$ identity matrix

$$\begin{aligned} (uvw) \sum_{i=1}^n m_i \dot{\underline{q}}^i &= (uvw) \begin{bmatrix} m^1 & m^2 & \dots & m^n \end{bmatrix} \begin{pmatrix} \dot{\underline{q}}^1 \\ \dot{\underline{q}}^2 \\ \vdots \\ \dot{\underline{q}}^n \end{pmatrix} \\ &= (uvw) [m^1 \ m^2 \ \dots \ m^n] [\Phi] \dot{\underline{n}} \end{aligned}$$

$$\underline{\omega}^T \sum_{i=1}^n m_i (\underline{r}^i + \underline{q}^i) \dot{\underline{q}}^i = \underline{\omega}^T [m_1 (\underline{r}^1 + \underline{q}^1) \ \dots \ m_n (\underline{r}^n + \underline{q}^n)] [\Phi] \dot{\underline{n}}$$

The kinetic energy can be written as the quadratic form $T = \frac{1}{2} \underline{U}^T [A] \underline{U}$ where $[A]$ is the coefficient matrix (symmetric) of the generalized accelerations appearing in the equations of motion (see Figure 3) and \underline{U} is the vector of non-holonomic velocities

$$\underline{U} = \left(uvw \begin{vmatrix} \underline{\omega}^T \\ \dot{\underline{n}}^T \end{vmatrix} \right)^T$$

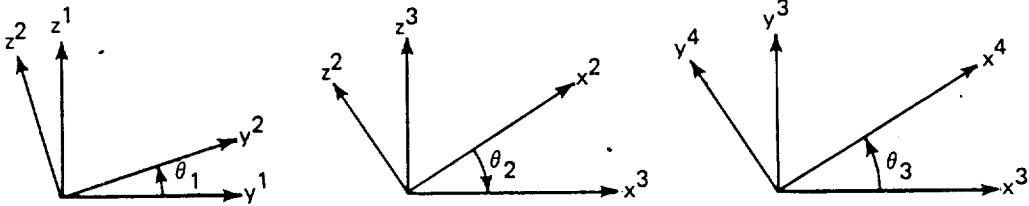
Since $[I_b]$ is positive definite, an inspection of the initial expression for T reveals that $T \geq 0$ for all \underline{U} . If $T = 0$ then $\vec{v}_b = \underline{\omega} = \underline{v}^i = \underline{0}$ ($i = 1, 2, \dots, n$). But $\vec{v}_b = \underline{0} = \underline{\omega}$ implies $(uvw) = \underline{0}$ and $\underline{v}^i = \underline{0} = (uvw) = \underline{\omega}$ implies $\dot{\underline{q}} = \underline{0}$. Hence $[\Phi] \dot{\underline{n}} = \underline{0}$. Since the columns of $[\Phi]$ are linearly independent we must have $\dot{\underline{n}} = \underline{0}$ also. In other words, $T = 0$ if and only if $\underline{U} = \underline{0}$. This argument proves that $[A]$ is positive definite and consequently nonsingular (see Chapter 8 where we require $[A]^{-1}$).

Note that if we replace the rigid body by a particle then $\underline{s} = \underline{0}$ and $[I_b] = [0]$. We still have $T \geq 0$ but if $T = 0$ we can only argue that $(uvw) = \underline{0}$. We can have $T = 0$ for nonzero $\underline{\omega}$ and $\dot{\underline{n}}$ as long as $\underline{\omega} \times (\underline{r}^i + \underline{q}^i) + \dot{\underline{q}}^i = 0$ ($i = 1, 2, \dots, n$). Thus for this later case $[A]$ is positive semi-definite. In particular $[A]$ will be singular. The situation here can be understood by simply enumerating the degrees of freedom involved. Originally we had a system consisting of a rigid body and n particles: $(6 + 3n)$ degrees of freedom. The number of dynamic equations was also $(6 + 3n)$. When degenerating the rigid body to a particle we have a system of $(n + 1)$ particles: $(3n + 3)$ degrees of freedom. However, when we retain the same equations of motion as in the original case $(6 + 3n)$ there will clearly be a redundancy present. Indeed, this explains why $[A]$ is singular for the degenerate case. Consequently, we cannot use the equations developed here for a system composed solely of particles; at least not without modification.

CHAPTER 7

KINEMATICAL RELATIONSHIPS

Let the transformation from the inertial frame $\{x^1, y^1, z^1\}$ to the body frame $\{x^4, y^4, z^4\}$ be arrived at by a sequence of three Euler angles $\theta_1, \theta_2, \theta_3$ as depicted below.



$$R^{12} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & \cos \theta_1 & -\sin \theta_1 \\ 0 & \sin \theta_1 & \cos \theta_1 \end{pmatrix}$$

$$R^{23} = \begin{pmatrix} \cos \theta_2 & 0 & \sin \theta_2 \\ 0 & 1 & 0 \\ -\sin \theta_2 & 0 & \cos \theta_2 \end{pmatrix}$$

$$R^{34} = \begin{pmatrix} \cos \theta_3 & -\sin \theta_3 & 0 \\ \sin \theta_3 & \cos \theta_3 & 0 \\ 0 & 0 & 1 \end{pmatrix}$$

$[R^{ij}]$ is the transformation matrix from frame 'j' to frame 'i'

Concatenating transformations, $[R^{14}] = [R^{12}][R^{23}][R^{34}]$

$$[R^{14}] = \begin{pmatrix} \cos\theta_2 \cos\theta_3 & -\cos\theta_2 \sin\theta_3 & \sin\theta_2 \\ \cos\theta_1 \sin\theta_3 + \sin\theta_1 \sin\theta_2 \cos\theta_3 & \cos\theta_1 \cos\theta_3 - \sin\theta_1 \sin\theta_2 \sin\theta_3 & -\sin\theta_1 \cos\theta_2 \\ \sin\theta_1 \sin\theta_3 - \cos\theta_1 \sin\theta_2 \cos\theta_3 & \sin\theta_1 \cos\theta_3 + \cos\theta_1 \sin\theta_2 \sin\theta_3 & \cos\theta_1 \cos\theta_2 \end{pmatrix} \quad (7-1)$$

We next derive the relationship between the body frame angular velocity $\underline{\omega}$ (expressed in body coordinates) and the Euler rates $\dot{\theta}_1, \dot{\theta}_2, \dot{\theta}_3$. Let $\{\underline{i}_p, \underline{j}_p, \underline{k}_p\}$ be the set of unit vectors along the axes of frame 'p' ($p = 1, 2, 3, 4$).

$$\vec{\omega} = \dot{\theta}_1 \vec{i}_1 + \dot{\theta}_2 \vec{j}_2 + \dot{\theta}_3 \vec{k}_3$$

To express $\vec{\omega}$ in the body frame, we must use the representation of the unit vectors in frame 4. With the aid of the transformations listed above we arrive at

$$\underline{\omega} = \begin{pmatrix} \dot{\theta}_1 \cos\theta_2 \cos\theta_3 + \dot{\theta}_2 \sin\theta_3 \\ \dot{\theta}_2 \cos\theta_3 - \dot{\theta}_1 \cos\theta_2 \sin\theta_3 \\ \dot{\theta}_3 + \dot{\theta}_1 \sin\theta_2 \end{pmatrix} \quad (\text{in body frame})$$

This system can be inverted to yield

$$\begin{pmatrix} \dot{\theta}_1 \\ \dot{\theta}_2 \\ \dot{\theta}_3 \end{pmatrix} = \begin{bmatrix} \frac{\cos\theta_3}{\cos\theta_2} & \frac{-\sin\theta_3}{\cos\theta_2} & 0 \\ \sin\theta_3 & \cos\theta_3 & 0 \\ -\cos\theta_3 \tan\theta_2 & \sin\theta_3 \tan\theta_2 & 1 \end{bmatrix} \underline{\omega} \quad (\cos\theta_2 \neq 0) \quad (7-2)$$

CHAPTER 8

EQUATIONS OF MOTION — FIRST ORDER FORM

The assembled motion equations (Figure 3) can be written as

$$[A(t)] \begin{bmatrix} \dot{\underline{u}} \\ \dot{\underline{v}} \\ \dot{\underline{w}} \\ \dot{\underline{e}} \\ \dot{\underline{n}} \end{bmatrix} = \underline{F} + \underline{U} - \begin{bmatrix} \underline{0} \\ \underline{0} \\ \omega_1^2 \eta_1 \\ \omega_2^2 \eta_2 \\ \vdots \\ \omega_t^2 \eta_t \end{bmatrix} \quad (8-1)$$

where

$$\underline{F} = \begin{bmatrix} \sum_{i=0}^n \underline{f}^i \\ \underline{r}^0 + \sum_{i=1}^n (\underline{r}^i + \underline{q}^i) \times \underline{f}^i \\ [\Phi]^T \begin{pmatrix} \underline{f}^1 \\ \underline{f}^2 \\ \vdots \\ \underline{f}^n \end{pmatrix} \end{bmatrix} \quad (8-2)$$

and

$$\underline{U} = \begin{bmatrix} \underline{u}_t \\ \hline \underline{u}_r \\ \hline [\phi]^T \underline{u}_v \end{bmatrix} \quad (8-3)$$

Let R_x, R_y, R_z be the components of the inertial position vector of O_g (origin of body frame) resolved along inertial axes and $[\Gamma]$ denote the matrix in Eq. (7-2). The kinematic relationships can now be written

$$\begin{pmatrix} \dot{R}_x \\ \dot{R}_y \\ \dot{R}_z \end{pmatrix} = [R^{14}] \begin{pmatrix} u \\ v \\ w \end{pmatrix}, \quad \begin{pmatrix} \dot{\theta}_1 \\ \dot{\theta}_2 \\ \dot{\theta}_3 \end{pmatrix} = [\Gamma] \underline{\omega}$$

Define $[\Omega^2] = \text{diag}(\omega_1^2, \omega_2^2, \dots, \omega_t^2)$. The state vector \underline{y} is defined to be

$$\underline{y} = (R_x R_y R_z \theta_1 \theta_2 \theta_3 \underline{n}^T \underline{uvw} \underline{\omega}^T \underline{\dot{n}}^T)^T \quad (8-4)$$

The equations of motion written in first order form are

$$\frac{d}{dt} \underline{y} = \begin{bmatrix} [R^{14}] \begin{pmatrix} u \\ v \\ w \end{pmatrix} \\ [\Gamma] \underline{\omega} \\ \underline{\dot{n}} \\ A^{-1} \left[\underline{F} + \underline{U} - \begin{pmatrix} 0 \\ [\Omega^2] \underline{n} \end{pmatrix} \right] \end{bmatrix} \quad (8-5)$$

This system of $(2t + 12)$ first order equations can be integrated numerically with appropriate initial conditions.

CHAPTER 9

DIGITAL SIMULATION

This chapter is concerned with the FORTRAN computer program which implements and numerically integrates the complete set of first order ordinary differential equations presented in Chapter 8, Eq. (8-5). A description of the main program, its subroutines, and the input data is given. An annotated flowchart of the program is given in Figure 4 and a complete listing of the program and its subroutines is provided in Appendix A. An example of the input data for a sample vehicle is provided in Appendix B. The code is liberally commented throughout and in most instances the FORTRAN variable names are mnemonically similar to the corresponding analytical quantities. Virtually all computations involving real number quantities are performed in (IBM) double precision. External subroutines from the double precision IMSL library⁽⁸⁾ are used to perform certain standard computations. IMSL subroutine "EIGRS" is used for eigenvalue/eigenvector extraction and subroutine "LEQTLP" is used to solve simultaneous linear equations. In addition, IMSL subroutine "USPLT" is used to generate time history graphs of selected elements of the vector

$$\{R_x R_y R_z \theta_1 \theta_2 \theta_3 \dot{q}^1 \dots \dot{q}^n uvw \omega_1 \omega_2 \omega_3 \dot{q}^1 \dots \dot{q}^n\} \quad (9-1)$$

via the line printer.

Throughout the program deformation dependent terms are arranged and computed hierarchically as quantities involving structural deflections to the first and second degree. Similarly the nonlinear kinematic terms

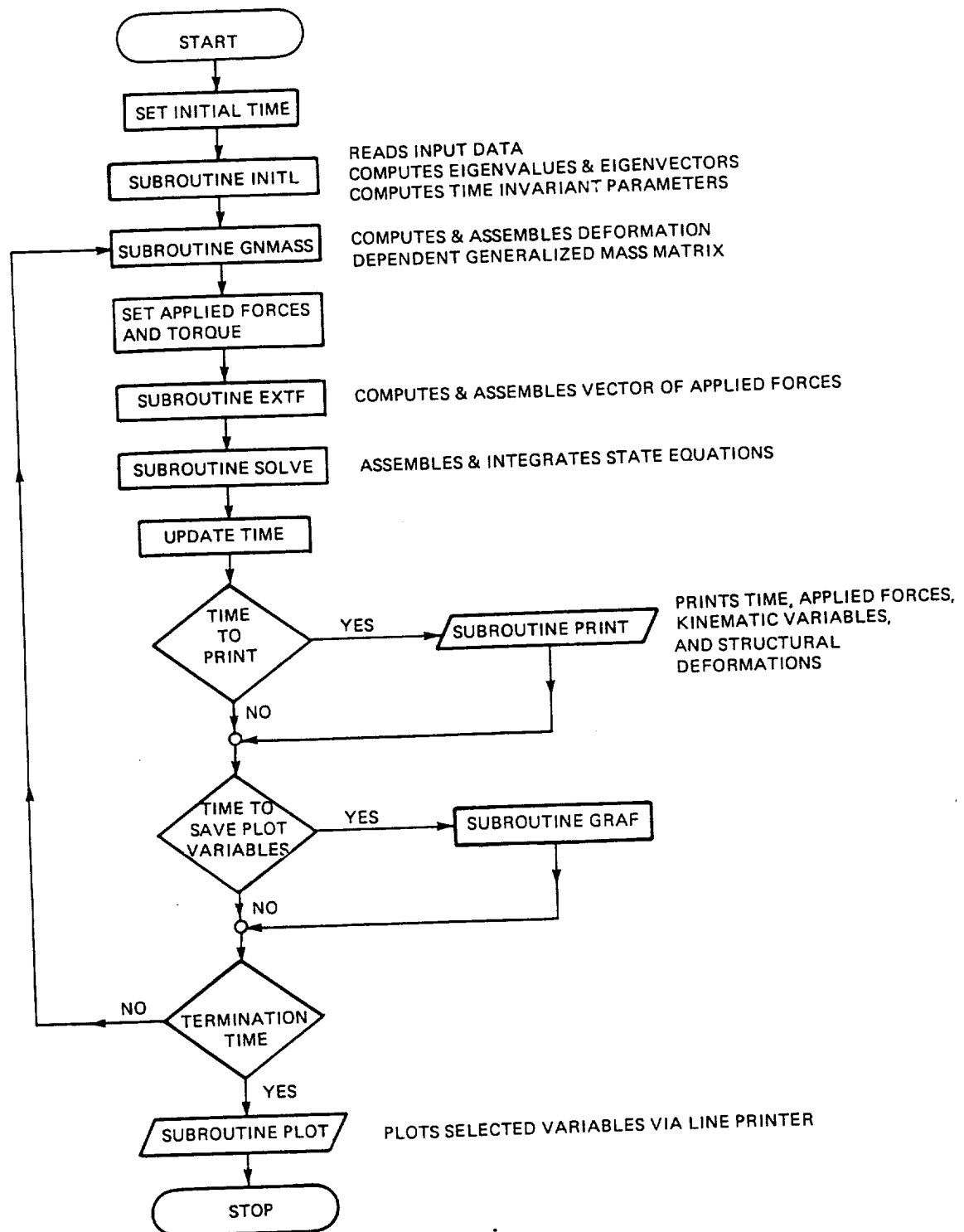


Figure 4. Program flowchart.

are organized into the three categories of Section 4.2 with the contributions of each group of terms being computed independently. This partitioned structure of the computations provides the capability to assess the influence of these higher order terms on the final solution and upon such analysis bypass those deemed negligible.

Main Program

The main program is simply an executive module which calls the appropriate subroutines in the proper order. The reader will note that if external forces are required, these must be explicitly coded either in the main program or as individual subroutines. If the external forces are time dependent, it is essential that they be recomputed prior to each call to subroutine "EXTF" (see comments in main program). For the system in Figure 1 the external excitation is accommodated via the three arrays: F_0 ,* TAU_0 , FP .

F_0 — sum of external forces on rigid body

TAU_0 — sum of external moments on rigid body taken about body frame origin.

F_0 and TAU_0 are three-dimensional vectors whose elements refer to components along body frame axes.

$FP(I,J)$ — is the I^{th} component of the external force acting upon particle J in the appendage ($I = 1,2,3$; $J = 1,2,\dots,N$).

The external forces for each of the "N" particles comprising the appendage are resolved along body axes.

Subroutines

Subroutine INITL reads in all program input data and performs consistency checks. Selected input data is echo printed. The eigenvalues

* " \emptyset " denotes the number zero.

and eigenvectors of the standard symmetric eigenvalue problem given by Eq. (5-2) are computed via a call to IMSL subroutine EIGRS. The eigenvectors are then transformed to those corresponding to Eq. (5-1). All time-invariant terms of the generalized mass matrix of Figure 3 are computed. Finally, the initial conditions on the particle displacements, modal coordinates, and the respective time derivatives are set.

Subroutine GNMASS computes and assembles the deformation dependent generalized mass matrix of Figure 3.

Subroutine EXTFF computes and assembles the generalized force vector \underline{F} of Eq. (8-2).

Subroutine NLKT computes and assembles the vector of nonlinear kinematic terms of Eq. (8-3).

Subroutine SOLVE computes the transformation matrices given by Eq. (7-1) and (7-2). The set of simultaneous equations given by Eq. (8-1) are solved via a call to IMSL subroutine LEQT1P. The state vector Eq. (8-4) is assembled and its time derivative, Eq. (8-5), evaluated. The value of the state vector is advanced one time step via a call to subroutine ODESLV.

Subroutine ODESLV integrates the state equation, Eq. (8-5), using the Adams method with third order differences.

Subroutine PRINT is executed only at print-time intervals specified in the input (see below). When called, the subroutine prints the time, force, and torque on the rigid body, applied forces on the particles and all the variables of the vector given in Eq. (9-1).

Entry point GRAF in subroutine PRINT stores selected variables for plotting at a specified time interval (see namelist items DTG and IPLOT below).

Program Input Data

Program input data is read in during execution of subroutine INITL. Input is achieved through four READ-NAMELIST combinations and a single unformatted READ of the stiffness matrix. It is worth noting that while the code given in Appendix A requires the stiffness matrix (described in Section 3.1) and from this and the appendage mass matrix (assembled internally) computes the constrained appendage eigenvalues and eigenvectors, it could be modified to read in the appropriate eigenvalues/eigenvectors directly. The four NAMELIST inputs are defined below, and their use illustrated in Appendix B.

- (1) NAMELIST/INPUT/MØ, N, MASS, RM, IØ, S, NT; contains all mass and geometry data as well as the number of modes to be retained.
 - MØ = mass of rigid body (real)
 - N = number of particles (integer)
 - MASS = masses of particles 1 through N (real $N \times 1$ array)
 - RM = position vectors of particles 1 through N prior to deformation, expressed in body frame (real $3 \times N$ array)
 - IØ = inertia matrix of the rigid body with respect to a frame located at the rigid body mass center with axes parallel to body frame (real 3×3 array)
 - S = position vector from body frame origin to mass center of rigid body expressed in body frame (real 3×1 array)
 - NT = number of modes to be retained; modes 1 through NT are used (integer)

- (2) NAMELIST/KIN/UVW, OMEGA, R, THETA: contains initial conditions for kinematic variables.

UVW = initial velocity vector of body frame origin, expressed in body frame coordinates (real 3×1 array)

OMEGA = initial angular velocity vector of body frame with respect to inertial frame, components expressed in body frame (real 3×1 array)

R = initial inertial position vector of body frame origin, components expressed in inertial frame (real 3×1 array)

THETA = initial 1-2-3 Euler angles of body frame with respect to inertial frame (real 3×1 array)

- (3) NAMELIST/RUN/DT, TSTOP, DTP, DTG: contains numerical integration parameters and print and plot time intervals.

DT = integration time step in seconds (real)

TSTOP = integration termination time in seconds (real)

DTP = print output time interval in seconds; output printed every DTP seconds (real)

DTG = plot output time interval in seconds; selected variables plotted every DTG seconds (real)

- (4) NAMELIST/PLOT/IPLOT: specifies which elements of vector in Eq. (9-1) are to be plotted via line printer.

IPLOT = integer array with the integers corresponding to those elements of the vector in Eq. (9-1) that are to be plotted versus time (see sample use in Appendix B).

APPENDIX A

FORTTRAN PROGRAM LISTING

```

LEVEL 2.3.0 (JUNE 78)                                05/350  FORTTRAN H EXTENDED          DATE 02.271/12.43.05      PAGE 1

REQUESTED OPTIONS: NOOBJ,TERM,,NOXREF,,NOMAP,,,NAME(MAIN),AD(NONE),OPT(0),,,FLAG(I),SIZE(304K),LC(60),

OPTIONS IN EFFECT: NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOCOLL(MORE)
SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORNAT CCGTHT NOXREF NOALC NOANSF TERM IBM FLAG(I)

C
C *****00000100
C *****00000200
C *THIS PROGRAM SOLVES THE EQUATIONS OF MOTION OF A VEHICLE 00000300
C *CONSISTS OF A RIGID BASE WITH AN ATTACHED FLEXIBLE APPENDAGE. 00000400
C *THE APPENDAGE IS IDEALIZED AS A COLLECTION OF PARTICLES CONNECTED 00000500
C *BY MASSLESS ELASTIC STRUCTURE. IN ADDITION TO EXTERNAL FORCES ACTING 00000600
C *UPON EACH OF THE PARTICLES, A FORCE AND TORQUE ARE ACCOMMODATED AT 00000700
C *THE GRAPPLE FIXTURE CORRESPONDING TO THE ORIGIN OF BODY FRAME.. 00000800
C *(WRITTEN BY JOEL STORCH & STEPHEN GATES C.S.D.L. BASED UPON 00000900
C * C.S.D.L. REPORT R21502 SEPT. 1982) 00001000
C *****00001100
C *****00001200
C *****00001300
C
C NOTE: ARRAYS ARE DIMENSIONED TO ACCOMMODATE A MAXIMUM OF 50 PARTICLES00001400
C *****00001500
C *****00001600
ISN 0002      IMPLICIT REAL*8(A-H,O-Z) 00001700
ISN 0003      DIMENSION F0(3),TAU0(3),FP(3,50) 00001800
ISN 0004      COMMON /TIME/ DT,TSTOP,DTP,DTG 00001900
ISN 0005      T=0.0 00002000
C INPUT PROGRAM DATA AND CALCULATE ALL TIME INVARIANT PARAMETERS. 00002100
C *****00002200
ISN 0006      CALL INITL 00002300
ISN 0007      TP=DTP 00002400
ISN 0008      TG=DTG 00002500
C *****00002600
C CALCULATE GENERALIZED MASS MATRIX 00002700
C *****00002800
ISN 0009      10 CALL GNMAT 00002900
C *****00003000
C INPUT VALUES REQUIRED FOR 'EXTF' 00003100
C (ALL VECTORS EXPRESSED IN BODY FRAME) 00003200
C *****00003300
C F0 - EXTERNAL FORCE ON M0 (AT ORIGIN OF BODY FRAME) 00003400
C TAU0 - EXTERNAL TORQUE AT LOCATION OF BODY FRAME ORIGIN. 00003500
C FP - VECTOR OF EXTERNAL FORCES ON PARTICLES 1,2,...,N. 00003600
C *****00003700
ISN 0010      CALL EXTF(F0,TAU0,FP) 00003800
C *****00003900
C CALCULATE NON-LINEAR KINEMATIC TERMS 00004000
C *****00004100
ISN 0011      CALL NLKT 00004200
C *****00004300
C INTEGRATE EQUATIONS OF MOTION 00004400
C *****00004500
ISN 0012      CALL SOLVE 00004600
ISN 0013      T=T+DT 00004700
ISN 0014      IF(T .LT. TP) GO TO 15 00004800

```

LEVEL 2.3.0 (JUNE 78) MAIN OS/360 FORTRAN H EXTENDED DATE 82.246/12.21.44 PAGE 2

ISN 0026		CALL PRINT(T,F0,TAU0,FP)	00005400
ISN 0027		TP=TP+DTP	00005500
ISN 0028	15	IF(T .LT. TG) GO TO 20	00005600
ISN 0030		CALL GRAF(T)	00005700
ISN 0031		TG=TG+DTG	00005800
ISN 0032	20	IF(T .LT. TSTOP) GO TO 10	00005900
ISN 0034		CALL PLOT	00006000
ISN 0035		STOP	00006100
ISN 0036		END	00006200

*OPTIONS IN EFFECT*NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)

*OPTIONS IN EFFECT*SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

STATISTICS SOURCE STATEMENTS = 35, PROGRAM SIZE = 2002, SUBPROGRAM NAME = MAIN

STATISTICS NO DIAGNOSTICS GENERATED

***** END OF COMPILATION *****

280K BYTES OF CORE NOT USED

REQUESTED OPTIONS: NOOBJ,TERM,,NOXREF,,NOMAP,,,NAME(MAIN),AD(NONE),OPT(0),,,FLAG(I),SIZE(384K),LC(60),

OPTIONS IN EFFECT: NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTODBL(NONE)
SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTHT NOXREF NOALC NOANSF TERM IBM FLAG(I)

ISN 0002	SUBROUTINE SKEW(V,A)	00006300
	C	00006400
	C THIS SUBROUTINE CREATES THE SKEW SYMMETRIC MATRIX CORRESPONDING	00006500
	C TO THE VECTOR "V".	00006600
	C	00006700
ISN 0003	REAL*8 V,A	00006800
ISN 0004	DIMENSION V(3),A(3,3)	00006900
ISN 0005	A(1,1)=0.0	00007000
ISN 0006	A(1,2)=-V(3)	00007100
ISN 0007	A(1,3)=V(2)	00007200
ISN 0008	A(2,1)=V(3)	00007300
ISN 0009	A(2,2)=0.0	00007400
ISN 0010	A(2,3)=-V(1)	00007500
ISN 0011	A(3,1)=-V(2)	00007600
ISN 0012	A(3,2)=V(1)	00007700
ISN 0013	A(3,3)=0.0	00007800
ISN 0014	RETURN	00007900
ISN 0015	END	00008000

*OPTIONS IN EFFECT*NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTODBL(NONE)

*OPTIONS IN EFFECT*SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTHT NOXREF NOALC NOANSF TERM IBM FLAG(I)

STATISTICS SOURCE STATEMENTS = 14, PROGRAM SIZE = 388, SUBPROGRAM NAME = SKEW

STATISTICS NO DIAGNOSTICS GENERATED

***** END OF COMPILATION *****

280K BYTES OF CORE NOT USED

LEVEL 2.3.0 (JUNE 78)

OS/360 FORTRAN H EXTENDED

DATE 82.246/12.21.46

PAGE 1

REQUESTED OPTIONS: NOOBJ,TERM,,NOXREF,,NOMAP,,,NAME(MAIN),AD(NONE),OPT(0),,,FLAG(I),SIZE(384K),LC(60),

OPTIONS IN EFFECT: NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTODBL(NONE)
SOURCE EBCDIC NOLIST NOOECCK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

ISN 0002		SUBROUTINE CROSS(A,B,C)	00008100
	C		00008200
	C	THIS SUBROUTINE CALCULATES THE VECTOR CROSS PRODUCT	00008300
	C	A X B =C	00008400
	C		00008500
			00008600
ISN 0003		REAL*8 A(3),B(3),C(3)	00008700
ISN 0004		C(1)=A(2)*B(3)-A(3)*B(2)	00008800
ISN 0005		C(2)=A(3)*B(1)-A(1)*B(3)	00008900
ISN 0006		C(3)=A(1)*B(2)-A(2)*B(1)	00009000
ISN 0007		RETURN	00009100
ISN 0008		END	

*OPTIONS IN EFFECT*NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTODBL(NONE)

*OPTIONS IN EFFECT*SOURCE EBCDIC NOLIST NOOECCK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

STATISTICS SOURCE STATEMENTS = 7, PROGRAM SIZE = 484, SUBPROGRAM NAME = CROSS

STATISTICS NO DIAGNOSTICS GENERATED

280K BYTES OF CORE NOT USED

***** END OF COMPILATION *****

REQUESTED OPTIONS: NOOBJ,TERM,,NOXREF,,NOMAP,,,NAME(MAIN),AD(NONE),OPT(0),,,FLAG(I),SIZE(384K),LC(60),

OPTIONS IN EFFECT: NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)
SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

```

ISN 0002      SUBROUTINE INITL                                00009200
C                                                     00009300
C THIS SUBROUTINE READS IN PROGRAM DATA AND CALCULATES ALL TIME 00009400
C INVARIANT PARAMETERS.                                00009500
C                                                     00009600
ISN 0003      IMPLICIT REAL*8(A-H,O-Z)                    00009700
ISN 0004      REAL*8 MASS,M0,K,MCO,INERT1,MRM,I0,MOS      00009800
ISN 0005      REAL*4 PLTDAT                                00009900
ISN 0006      DIMENSION MASS(50),RM(3,50),K(150,150),MCO(3),INERT1(3,3),
1 MRM(3,50),CHAT(3,3),A12(3,3),A23(3,150),A(156,156),Q(3,50),
1 QDOT(3,50),UVW(3),OMEGA(3),R(3),THETA(3),I0(3,3),S(3),MOS(3),
2 AV(11325),FREQ(150),EV(150,150),WK(150),IPL0T(42),PLTDAT(100,20),
3 WKM(3,150),ETA(150),ETAD(150)
ISN 0007      COMMON /CONST/ TM,MASS,MCO,MRM,CHAT,N,N3,N3P6,NT,NTP6,NO 00010500
ISN 0008      COMMON /AMAT/ A,A12,INERT1,A23              00010600
ISN 0009      COMMON /STATE/ R,THETA,Q,UVW,OMEGA,QDOT    00010700
ISN 0010      COMMON /GEOM/ RM                            00010800
ISN 0011      COMMON /TIME/ DT,TSTOP,DTP,DTG            00010900
ISN 0012      COMMON /RIGID/ MOS,I0,S                   00011000
ISN 0013      COMMON /PLOT/ PLTDAT,IPL0T,NP              00011100
ISN 0014      COMMON /MODCO/ ETA,ETAD                    00011200
ISN 0015      COMMON /MODES/ EV,FREQ                     00011300
ISN 0016      EQUIVALENCE(K(1,1),EV(1,1))               00011400
ISN 0017      NAMELIST /INPUT/ M0,N,MASS,RM,I0,S,NT      00011500
ISN 0018      NAMELIST /KIN/ UVW,OMEGA,R,THETA           00011600
ISN 0019      NAMELIST /RUN/ DT,TSTOP,DTP,DTG            00011700
ISN 0020      NAMELIST /PLOT/ IPL0T                      00011800
C                                                     00011900
C DESCRIPTION OF NAMELIST VARIABLES                    00012000
C "M0" IS THE MASS OF THE RIGID BASE                    00012100
C                                                     00012200
C "N" IS THE NUMBER OF PARTICLES THAT COMPRISE THE FLEXIBLE APPENDAGE 00012300
C                                                     00012400
C "MASS" CONTAINS THE MASSES OF PARTICLES 1 TO N        00012500
C                                                     00012600
C "RM" CONTAINS THE POSITION VECTORS OF PARTICLES 1 THRU N IN THE 00012700
C UNDEFORMED STATE EXPRESSED IN THE BODY FRAME.        00012800
C                                                     00012900
C "I0" IS THE INERTIA MATRIX OF THE RIGID BASE WITH RESPECT TO A 00013000
C FRAME LOCATED AT THE MASS CENTER OF THE BASE AND PARALLEL 00013100
C TO THE BODY FIXED AXIS SYSTEM.                        00013200
C                                                     00013300
C "S" IS THE VECTOR FROM THE BODY FRAME ORIGIN TO THE MASS CENTER 00013400
C OF THE RIGID BODY.                                    00013500
C                                                     00013600
C "NT" NUMBER OF RETAINED MODES IN APPENDAGE VIBRATION 00013700
C                                                     00013800
C "UVW" IS THE INITIAL VELOCITY OF THE BODY FRAME ORIGIN EXPRESSED 00013900
C IN BODY COORDINATES.                                  00014000
C                                                     00014100
C "OMEGA" IS THE INITIAL ANGULAR VELOCITY OF THE BODY FRAME EXPRESSED 00014200
C IN BODY COORDINATES.                                  00014300
C                                                     00014400

```

```

C
C
C "R" IS THE INITIAL INERTIAL POSITION VECTOR OF THE BODY FRAME
C ORIGIN
C
C "THETA" IS THE INITIAL SET OF ATTITUDE ANGLES FOR THE BODY FRAME
C
C "DT" IS THE INTEGRATION TIME STEP
C
C "TSTOP" IS THE TERMINAL TIME FOR THE SIMULATION
C
C "DTP" IS THE TIME INTERVAL BETWEEN PRINTOUTS
C
C "DTG" IS THE TIME INTERVAL BETWEEN PLOTTED POINTS
C
C "IPILOT" IS AN ARRAY INDICATING VARIABLES TO BE PLOTTED.
C NUMBERING CORRESPONDS TO LOCATION IN STATE VECTOR.
C POINTS ARE PLOTTED EVERY "DTG" SECONDS.
C
ISN 0021 READ(5,INPUT)
ISN 0022 IF(NT .LE. 3*N) GO TO 8
ISN 0024 WRITE(6,112) NT,N
ISN 0025 STOP
ISN 0026 8 WRITE(6,100) M0
ISN 0027 DO 10 I=1,N
ISN 0028 10 WRITE(6,101) I,MASS(I),(RM(J,I),J=1,3)
ISN 0029 100 FORMAT(1H1,20X,'MASS OF RIGID BASE=',E12.4,' SLUGS',//,1H ,T10,
1 'PARTICLE',T21,'MASS(SLUGS)',T41,'POSITION(FT.)',/)
ISN 0030 101 FORMAT(1H ,T12,I3,T21,F7.2,T34,3(F7.2,2X))
ISN 0031 WRITE(6,106) ((IO(I,J),J=1,3),I=1,3)
ISN 0032 WRITE(6,107) S,NT
ISN 0033 READ(5,KIN)
ISN 0034 WRITE(6,104) R,THETA
ISN 0035 103 FORMAT(1H0,5X,'INITIAL VELOCITY=',3F7.2,3X,'FT/SEC',4X,
1 'INITIAL ANGULAR VELOCITY=',3F7.2,' DEG/SEC')
ISN 0036 WRITE(6,103) UVM,OMEGA
ISN 0037 104 FORMAT(1H0,5X,'INITIAL POSITION=',3F7.2,3X,'FT',4X,
1 'INITIAL ATTITUDE=',3F8.3,' DEG')
ISN 0038 READ(5,RUN)
ISN 0039 WRITE(6,105) DT,TSTOP,DTP,DTG
ISN 0040 READ(5,PLOT)
ISN 0041 II=0
ISN 0042 DO 108 I=1,42
ISN 0043 IF(IPILOT(I) .EQ. 0) GO TO 109
ISN 0045 108 II=II+1
ISN 0046 109 IF(II .EQ. 0) GO TO 111
ISN 0048 WRITE(6,110) (IPILOT(I),I=1,II)
ISN 0049 110 FORMAT(1H0,5X,'VARIABLES PLOTTED',4X,42(I2,1X))
ISN 0050 105 FORMAT(1H0,5X,'TIME STEP=',E12.4,' SEC',3X,'TERMINATION TIME=',
1 E12.4,' SEC',2X,'PRINT INTERVAL=',E12.4,' SEC', 'PLOT INTERVAL=',
2 E12.4,' SEC')
ISN 0051 112 FORMAT(1H0,3X,I2,' MODES REQUESTED',2X,I3,' PARTICLES IN MODEL')
ISN 0052 106 FORMAT(1H0,T15,'INERTIA MATRIX OF RIGID BODY(SLUG FT**2)',
1 //,(T15,3E13.5))
ISN 0053 107 FORMAT(1H0,15X,'S=',3E13.5,' FT',3X,I2,' CONSTRAINED APPENDAGE MODE',
1S RETAINED')
C
C CHANGE ANGULAR VELOCITY & ATTITUDE TO RADIAN MEASURE

```



```

C
ISN 0054 111 DTR=DATAN(1.000)/45.
ISN 0055 DO 15 I=1,3
ISN 0056 OMEGA(I)=DTR*OMEGA(I)
ISN 0057 15 THETA(I)=DTR*THETA(I)
C
C TH - TOTAL BODY MASS
C
ISN 0058 TH=M0
ISN 0059 DO 20 I=1,N
ISN 0060 20 TH=TH+MASS(I)
ISN 0061 DO 30 J=1,3
ISN 0062 M0S(J)=M0*S(J)
ISN 0063 30 M0(J)=0.0
ISN 0064 DO 40 I=1,N
ISN 0065 DO 45 J=1,3
ISN 0066 MRM(J,I)=MASS(I)*RM(J,I)
ISN 0067 45 M0(J)=M0(J)+MRM(J,I)
ISN 0068 40 CONTINUE
ISN 0069 DO 42 I=1,3
ISN 0070 42 M0(I)=M0(I)+M0S(I)
ISN 0071 N3=3*N
ISN 0072 N3P6=N3+6
ISN 0073 NTP6=NT+6
ISN 0074 NO=2*NTP6
ISN 0075 NP=0
C
C READ IN STIFFNESS MATRIX
C
ISN 0076 READ(8) NDOF,((K(I,J),J=1,NDOF),I=1,NDOF)
ISN 0077 IF(NDOF .EQ. N3) GO TO 400
ISN 0079 WRITE(6,102) N,NDOF
ISN 0080 STOP
ISN 0081 400 CONTINUE
ISN 0082 102 FORMAT(1H0,10X,'INCONSISTENT DATA',2X,I3,' PARTICLES',2X,I3,
1 ' DEGREES OF FREEDOM IN STIFFNESS MATRIX')
C
C GET CONSTRAINED FREQUENCIES AND MODE SHAPES OF APPENDAGE
C
ISN 0083 . L=1
ISN 0084 DO 300 J=1,N3
ISN 0085 LC=1+J/3
ISN 0086 IF( (J-3*(J/3)) .EQ. 0) LC=LC-1
ISN 0088 DO 300 I=1,J
ISN 0089 LR=1+I/3
ISN 0090 IF( (I-3*(I/3)) .EQ. 0) LR=LR-1
ISN 0092 AV(L)=K(I,J)/DSQRT(MASS(LR)*MASS(LC))
ISN 0093 L=L+1
ISN 0094 300 CONTINUE
ISN 0095 CALL EIGRS(AV,N3,1,FREQ,EV,150,WK,IER)
ISN 0096 IF(IER .EQ. 0) GO TO 310
ISN 0098 WRITE(6,301) IER
ISN 0099 301 FORMAT(1H0,10X,'ERROR FROM IMSL ROUTINE "EIGRS" ERROR CODE=',I4)
ISN 0100 STOP
C
C TRANSFORM EIGENVECTORS
C
ISN 0101 310 DO 311 L=1,N

```

```

00020300
00020400
00020500
00020600
00020700
00020800
00020900
00021000
00021100
00021200
00021300
00021400
00021500
00021600
00021700
00021800
00021900
00022000
00022100
00022200
00022300
00022400
00022500
00022600
00022700
00022800
00022900
00023000
00023100
00023200
00023300
00023400
00023500
00023600
00023700
00023800
00023900
00024000
00024100
00024200
00024300
00024400
00024500
00024600
00024700
00024800
00024900
00025000
00025100
00025200
00025300
00025400
00025500
00025600
00025700
00025800
00025900
00026000

```

```

ISN 0102      C1=DSQRT(MASS(L))
ISN 0103      DO 311 I=1,3
ISN 0104      IR=3*(L-1)+I
ISN 0105      DO 311 J=1,N3
ISN 0106      311 EV(IR,J)=EV(IR,J)/C1
ISN 0107      DO 320 I=1,N3
ISN 0108      PHZ=DSQRT(FREQ(I))/6.283185
ISN 0109      WRITE(6,321) I,PHZ,(EV(J,I),J=1,N3)
ISN 0110      321 FORMAT(1H0,3X,'MODE ',I2,' FREQUENCY=',E12.4,1X,'(HZ)',/,1H ,
1 'MODE SHAPE: ',(9(6I2.4,1X)))
ISN 0111      320 CONTINUE

C
C THE ROUTINE "EIGRS" RETURNS AN ORTHONORMAL SET OF EIGENVECTORS.
C THIS IS ESSENTIAL SINCE WE ASSUME IN THE DERIVATION THAT THE
C EIGENVECTORS(OF THE ORIGINAL GENERALIZED EIGENVALUE PROBLEM)
C ARE ORTHOGONAL WITH RESPECT TO MASS(STIFFNESS) AND NORMALIZED
C WITH RESPECT TO MASS.
C
C
C CALCULATE TIME INVARIANT PART OF INERTIA MATRIX "INERT1"
C AND "CMAT"
C
ISN 0112      DO 50 I=1,3
ISN 0113      DO 50 J=1,3
ISN 0114      INERT1(I,J)=0.0
ISN 0115      CMAT(I,J)=0.0
ISN 0116      50 CONTINUE
ISN 0117      DO 60 I=1,N
ISN 0118      XS=RM(1,I)**2
ISN 0119      YS=RM(2,I)**2
ISN 0120      ZS=RM(3,I)**2
ISN 0121      INERT1(1,1)=INERT1(1,1)+MASS(I)*(YS+ZS)
ISN 0122      INERT1(1,2)=INERT1(1,2)-MASS(I)*RM(1,I)*RM(2,I)
ISN 0123      INERT1(1,3)=INERT1(1,3)-MASS(I)*RM(1,I)*RM(3,I)
ISN 0124      INERT1(2,2)=INERT1(2,2)+MASS(I)*(XS+ZS)
ISN 0125      INERT1(2,3)=INERT1(2,3)-MASS(I)*RM(2,I)*RM(3,I)
ISN 0126      INERT1(3,3)=INERT1(3,3)+MASS(I)*(XS+YS)
ISN 0127      CMAT(1,1)=CMAT(1,1)+MASS(I)*XS
ISN 0128      CMAT(2,2)=CMAT(2,2)+MASS(I)*YS
ISN 0129      CMAT(3,3)=CMAT(3,3)+MASS(I)*ZS
ISN 0130      60 CONTINUE
ISN 0131      CMAT(1,2)=-INERT1(1,2)
ISN 0132      CMAT(1,3)=-INERT1(1,3)
ISN 0133      CMAT(2,3)=-INERT1(2,3)
ISN 0134      INERT1(1,1)=INERT1(1,1)+I0(1,1)+M0*(S(2)**2+S(3)**2)
ISN 0135      INERT1(1,2)=INERT1(1,2)+I0(1,2)-M0*S(1)*S(2)
ISN 0136      INERT1(1,3)=INERT1(1,3)+I0(1,3)-M0*S(1)*S(3)
ISN 0137      INERT1(2,2)=INERT1(2,2)+I0(2,2)+M0*(S(1)**2+S(3)**2)
ISN 0138      INERT1(2,3)=INERT1(2,3)+I0(2,3)-M0*S(2)*S(3)
ISN 0139      INERT1(3,3)=INERT1(3,3)+I0(3,3)+M0*(S(1)**2+S(2)**2)
ISN 0140      DO 70 I=1,3
ISN 0141      DO 70 J=1,3
ISN 0142      IF(I.LE. J) GO TO 70
ISN 0143      INERT1(I,J)=INERT1(J,I)
ISN 0144      CMAT(I,J)=CMAT(J,I)
ISN 0145      70 CONTINUE
ISN 0146      C
C CREATE TIME INVARIANT PORTIONS OF GENERALIZED MASS MATRIX "A"

```

```

00026100
00026200
00026300
00026400
00026500
00026600
00026700
00026800
00026900
00027000
00027100
00027200
00027300
00027400
00027500
00027600
00027700
00027800
00027900
00028000
00028100
00028200
00028300
00028400
00028500
00028600
00028700
00028800
00028900
00029000
00029100
00029200
00029300
00029400
00029500
00029600
00029700
00029800
00029900
00030000
00030100
00030200
00030300
00030400
00030500
00030600
00030700
00030800
00030900
00031000
00031100
00031200
00031300
00031400
00031500
00031600
00031700
00031800

```

```

C
C (1,2) PARTITION "A12"
C
ISN 0147      CALL SKEW(MC0,A12)
ISN 0148      DO 80 I=1,3
ISN 0149      DO 80 J=1,3
ISN 0150      IF(I.EQ. J) GO TO 80
ISN 0152      A12(I,J)=-A12(I,J)
ISN 0153      80 CONTINUE

C
C (2,3) PARTITION "A23"
C
ISN 0154      DO 90 I=1,N
ISN 0155      L=3*I-2
ISN 0156      90 CALL SKEW(MRM(1,I),WKM(1,L))
ISN 0157      DO 92 I=1,3
ISN 0158      DO 92 J=1,NT
ISN 0159      A23(I,J)=0.0
ISN 0160      DO 94 L=1,N3
ISN 0161      94 A23(I,J)=A23(I,J)+WKM(I,L)*EV(L,J)
ISN 0162      92 CONTINUE

C
C STORE CONSTANT PARTITIONS OF "A"
C
ISN 0163      DO 200 I=1,NTP6
ISN 0164      DO 200 J=1,NTP6
ISN 0165      A(I,J)=0.0
ISN 0166      200 CONTINUE

C
C CREATE (1,1) PARTITION
C
ISN 0167      DO 210 I=1,3
ISN 0168      DO 210 J=1,3
ISN 0169      IF(I.EQ. J) A(I,J)=TH
ISN 0171      210 CONTINUE

C
C CREATE (1,3) PARTITION
C
ISN 0172      DO 215 I=1,3
ISN 0173      DO 215 J=1,N3
ISN 0174      215 WKM(I,J)=0.0
ISN 0175      DO 220 L=1,M
ISN 0176      JS=3*L-2
ISN 0177      DO 221 I=1,3
ISN 0178      WKM(I,JS)=MASS(L)
ISN 0179      JS=JS+1
ISN 0180      221 CONTINUE
ISN 0181      220 CONTINUE
ISN 0182      DO 283 I=1,3
ISN 0183      DO 283 J=1,NT
ISN 0184      JP6=J+6
ISN 0185      A(I,JP6)=0.0
ISN 0186      DO 281 L=1,N3
ISN 0187      281 A(I,JP6)=A(I,JP6)+WKM(I,L)*EV(L,J)
ISN 0188      283 CONTINUE

C
C CREATE (3,3) PARTITION
C

```

```

00031900
00032000
00032100
00032200
00032300
00032400
00032500
00032600
00032700
00032800
00032900
00033000
00033100
00033200
00033300
00033400
00033500
00033600
00033700
00033800
00033900
00034000
00034100
00034200
00034300
00034400
00034500
00034600
00034700
00034800
00034900
00035000
00035100
00035200
00035300
00035400
00035500
00035600
00035700
00035800
00035900
00036000
00036100
00036200
00036300
00036400
00036500
00036600
00036700
00036800
00036900
00037000
00037100
00037200
00037300
00037400
00037500
00037600

```

LEVEL 2.3.0 (JUNE 78)

INITL

OS/360 FORTRAN H EXTENDED

DATE 82.246/12.21.47

PAGE 6

ISN 0189		DO 230 I=1,NT	00037700
ISN 0190	230	A(L+6,L+6)=1.	00037800
	C		00037900
	C	SET INITIAL DEFORMATION AND RATE TO ZERO	00038000
	C		00038100
			00038200
ISN 0191		DO 250 I=1,N	00038300
ISN 0192		DO 250 J=1,3	00038400
ISN 0193		Q(J,I)=0.0	00038500
ISN 0194		QDOT(J,I)=0.0	00038600
ISN 0195	250	CONTINUE	00038700
ISN 0196		DO 293 I=1,NT	00038800
ISN 0197		ETA(I)=0.0	00038900
ISN 0198		ETAD(I)=0.0	00039000
ISN 0199	293	CONTINUE	00039100
ISN 0200		RETURN	00039200
ISN 0201		END	

*OPTIONS IN EFFECT*NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(8384K) AUTOOBL(NONE)

*OPTIONS IN EFFECT*SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

STATISTICS SOURCE STATEMENTS = 200, PROGRAM SIZE = 103026, SUBPROGRAM NAME = INITL

STATISTICS NO DIAGNOSTICS GENERATED

232K BYTES OF CORE NOT USED

***** END OF COMPILATION *****

REQUESTED OPTIONS: NOOBJ,TERM,,NOXREF,,NOMAP,,,NAME(MAIN),AD(NONE),OPT(0),,,FLAG(I),SIZE(384K),LC(60),

OPTIONS IN EFFECT: NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTODBL(NONE)
SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTHT NOXREF NOALC NOANSF TERM IBM FLAG(I)

```

ISN 0002      SUBROUTINE GNMASS                                00039300
C                                                     00039400
C THIS SUBROUTINE CALCULATES THE GENERALIZED MASS MATRIX "A" 00039500
C                                                     00039600
ISN 0003      IMPLICIT REAL*8(A-H,O-Z)                      00039700
ISN 0004      REAL*8 MASS,MC0,MRM,INERT1,MQ,MC1,INERT2,INERT3 00039800
ISN 0005      DIMENSION MASS(50),MC0(3),MRM(3,50),CMAT(3,3),A(156,156),
1 A12(3,3),INERT1(3,3),A23(3,150),Q(3,50),MQ(3,50),MC1(3),
2 WM(3,3),INERT2(3,3),A23Q(3,150),INERT3(3,3),QDOT(3,50),UVH(3),
3 OMEGA(3),R(3),THETA(3)                                00040100
ISN 0006      COMMON /CONST/ TM,MASS,MC0,MRM,CMAT,N,N3,N3P6,NT,NTP6,NO 00040300
ISN 0007      COMMON /AMAT/ A,A12,INERT1,A23                00040400
ISN 0008      COMMON /STATE/ R,THETA,Q,UVH,OMEGA,QDOT       00040500
ISN 0009      COMMON /TDEPV/ MQ,MC1                        00040600
ISN 0010      COMMON /MODES/ EV(150,150),FREQ(150)          00040700
ISN 0011      DO 10 I=1,3                                    00040800
ISN 0012      MC1(I)=0.0                                     00040900
ISN 0013      DO 40 I=1,N                                    00041000
ISN 0014      DO 45 J=1,3                                    00041100
ISN 0015      MQ(J,I)=MASS(I)*Q(J,I)                       00041200
ISN 0016      45 MC1(J)=MC1(J)+MQ(J,I)                     00041300
ISN 0017      40 CONTINUE                                   00041400
C                                                     00041500
C COMPUTE "A" NEGLECTING TIME DEPENDENT(DEFORMATION DEPENDENT) TERMS 00041600
C                                                     00041700
ISN 0018      DO 50 J=1,3                                    00041800
ISN 0019      JP3=J+3                                        00041900
ISN 0020      DO 50 I=1,3                                    00042000
ISN 0021      50 A(I,JP3)=A12(I,J)                          00042100
ISN 0022      DO 60 I=1,3                                    00042200
ISN 0023      IP3=I+3                                        00042300
ISN 0024      DO 60 J=1,3                                    00042400
ISN 0025      JP3=J+3                                        00042500
ISN 0026      60 A(IP3,JP3)=INERT1(I,J)                     00042600
ISN 0027      DO 70 J=1,NT                                   00042700
ISN 0028      JP6=J+6                                        00042800
ISN 0029      DO 70 I=1,3                                    00042900
ISN 0030      IP3=I+3                                        00043000
ISN 0031      70 A(IP3,JP6)=A23(I,J)                         00043100
C                                                     00043200
C ADD IN FIRST ORDER DEFORMATION TERMS                    00043300
C                                                     00043400
ISN 0032      CALL SKEW(MC1,WM)                              00043500
ISN 0033      DO 80 J=1,3                                    00043600
ISN 0034      JP3=J+3                                        00043700
ISN 0035      DO 80 I=1,3                                    00043800
ISN 0036      80 A(I,JP3)=A(I,JP3)-WM(I,J)                  00043900
C                                                     00044000
C FIRST ORDER DEFORMATION TERMS IN INERTIA MATRIX - "INERT2" 00044100
C                                                     00044200
ISN 0037      DO 85 I=1,3                                    00044300
ISN 0038      DO 85 J=1,3                                    00044400
ISN 0039      85 INERT2(I,J)=0.0                            00044500

```

```

ISN 0040      DO 90 L=1,3
ISN 0041      SUM=0.0
ISN 0042      DO 100 I=1,N
ISN 0043      SUM=SUM+MRM(L,I)*Q(L,I)
ISN 0044      SUM=2.0*SUM
ISN 0045      LL=L+1
ISN 0046      DO 110 II=1,2
ISN 0047      IF(LL.GT. 3) LL=1
ISN 0049      INERT2(LL,LL)=INERT2(LL,LL)+SUM
ISN 0050      LL=LL+1
ISN 0051      110 CONTINUE
ISN 0052      90 CONTINUE
ISN 0053      SUM=0.0
ISN 0054      DO 120 I=1,N
ISN 0055      SUM=SUM-(MRM(1,I)*Q(2,I)+MRM(2,I)*Q(1,I))
ISN 0056      INERT2(1,2)=SUM
ISN 0057      SUM=0.0
ISN 0058      DO 130 I=1,N
ISN 0059      SUM=SUM-(MRM(1,I)*Q(3,I)+MRM(3,I)*Q(1,I))
ISN 0060      INERT2(1,3)=SUM
ISN 0061      SUM=0.0
ISN 0062      DO 140 I=1,N
ISN 0063      SUM=SUM-(MRM(2,I)*Q(3,I)+MRM(3,I)*Q(2,I))
ISN 0064      INERT2(2,3)=SUM
ISN 0065      DO 150 I=1,3
ISN 0066      DO 150 J=1,3
ISN 0067      IF(I.LE. J) GO TO 150
ISN 0069      INERT2(I,J)=INERT2(J,I)
ISN 0070      150 CONTINUE
ISN 0071      DO 160 I=1,3
ISN 0072      IP3=I+3
ISN 0073      DO 160 J=1,3
ISN 0074      JP3=J+3
ISN 0075      A(IP3,JP3)=A(IP3,JP3)+INERT2(I,J)
ISN 0076      DO 170 I=1,N
ISN 0077      L=3*I-2
ISN 0078      170 CALL SKEW(MQ(1,I),A23Q(1,L))
ISN 0079      DO 180 I=1,3
ISN 0080      IP3=I+3
ISN 0081      DO 180 J=1,NT
ISN 0082      JP6=J+6
ISN 0083      DO 185 L=1,N3
ISN 0084      A(IP3,JP6)=A(IP3,JP6)+A23Q(I,L)*EV(L,J)
ISN 0085      180 CONTINUE
C
C ADD IN SECOND ORDER DEFORMATION TERMS - "INERT3"
C
ISN 0086      DO 190 I=1,3
ISN 0087      DO 190 J=1,3
ISN 0088      INERT3(I,J)=0.0
ISN 0089      DO 200 L=1,3
ISN 0090      SUM=0.0
ISN 0091      DO 210 I=1,N
ISN 0092      SUM=SUM+MQ(L,I)*Q(L,I)
ISN 0093      LL=L+1
ISN 0094      DO 220 II=1,2
ISN 0095      IF(LL.GT. 3) LL=1
ISN 0097      INERT3(LL,LL)=INERT3(LL,LL)+SUM

```

```

00044600
00044700
00044800
00044900
00045000
00045100
00045200
00045300
00045400
00045500
00045600
00045700
00045800
00045900
00046000
00046100
00046200
00046300
00046400
00046500
00046600
00046700
00046800
00046900
00047000
00047100
00047200
00047300
00047400
00047500
00047600
00047700
00047800
00047900
00048000
00048100
00048200
00048300
00048400
00048500
00048600
00048700
00048800
00048900
00049000
00049100
00049200
00049300
00049400
00049500
00049600
00049700
00049800
00049900
00050000
00050100
00050200
00050300

```

ISN 0098		LL=LL+1	00050400
ISN 0099	220	CONTINUE	00050500
ISN 0100	200	CONTINUE	00050600
ISN 0101		SUM=0.0	00050700
ISN 0102		DO 240 I=1,N	00050800
ISN 0103	240	SUM=SUM-MQ(1,I)*Q(2,I)	00050900
ISN 0104		INERT3(1,2)=SUM	00051000
ISN 0105		SUM=0.0	00051100
ISN 0106		DO 250 I=1,N	00051200
ISN 0107	250	SUM=SUM-MQ(1,I)*Q(3,I)	00051300
ISN 0108		INERT3(1,3)=SUM	00051400
ISN 0109		SUM=0.0	00051500
ISN 0110		DO 260 I=1,N	00051600
ISN 0111	260	SUM=SUM-MQ(2,I)*Q(3,I)	00051700
ISN 0112		INERT3(2,3)=SUM	00051800
ISN 0113		DO 270 I=1,3	00051900
ISN 0114		DO 270 J=1,3	00052000
ISN 0115		IF(I .LE. J) GO TO 270	00052100
ISN 0117		INERT3(I,J)=INERT3(J,I)	00052200
ISN 0118	270	CONTINUE	00052300
ISN 0119		DO 280 I=1,3	00052400
ISN 0120		IP3=I+3	00052500
ISN 0121		DO 280 J=1,3	00052600
ISN 0122		JP3=J+3	00052700
ISN 0123	280	A(IP3,JP3)=A(IP3,JP3)+INERT3(I,J)	00052800
	C		00052900
ISN 0124	302	FORMAT(1H0,/,1H ,10X,'A MATRIX')	00053000
ISN 0125	301	FORMAT(1H0,2X,15(F7.2,1X))	00053100
ISN 0126		RETURN	00053200
ISN 0127		END	00053300

*OPTIONS IN EFFECT*NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)

*OPTIONS IN EFFECT*SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTHT NOXREF NOALC NOANSF TERM IBM FLAG(I)

STATISTICS SOURCE STATEMENTS = 126, PROGRAM SIZE = 8156, SUBPROGRAM NAME =GNMASS

STATISTICS NO DIAGNOSTICS GENERATED

***** END OF COMPILATION *****

256K BYTES OF CORE NOT USED

REQUESTED OPTIONS: NOOBJ,TERM,,NOXREF,,NOMAP,,,NAME(MAIN),AD(NONE),OPT(0),,,FLAG(I),SIZE(384K),LC(60),

OPTIONS IN EFFECT: NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)
SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

```

      ISN 0002      SUBROUTINE EXTF(F0,TAU0,FP)
C
C      THIS SUBROUTINE ASSEMBLES THE FORCE VECTOR "F" IN THE
C      MOTION EQUATIONS AND IS PARTITIONED AS: FORCES FOR BODY
C      TRANSLATION, FORCES FOR BODY ROTATION, AND FORCES FOR
C      PARTICLE TRANSLATION.
C
C      INPUT TO SUBROUTINE
C
C      F0 - EXTERNAL FORCE ON M0 (AT ORIGIN OF BODY FRAME)
C      TAU0 - EXTERNAL TORQUE AT LOCATION OF BODY FRAME ORIGIN.
C      FP - VECTOR OF EXTERNAL FORCES ON PARTICLES 1,2,...,N.
C      (ALL VECTORS EXPRESSED IN BODY FRAME)
C
      IMPLICIT REAL*8(A-H,O-Z)
      REAL*8 MASS,MC0,MRM
      DIMENSION F0(3),TAU0(3),FP(3,50),Q(3,50),RM(3,50),MASS(50),MC0(3),
1 MRM(3,50),CHAT(3,3),F(156),SUM(3),WV(3),WV1(3),QDOT(3,50),UVH(3),
2 OMEGA(3),R(3),THETA(3),WV2(150),PHI(150,150),WS(150)
      COMMON /STATE/ R,THETA,Q,UVH,OMEGA,QDOT
      COMMON /GEOM/ RM
      COMMON /CONST/ TH,MASS,MC0,MRM,CHAT,N,N3,N3P6,NT,NTP6,NO
      COMMON /FORCE/ F
      COMMON /MODES/ PHI,WS
      DO 10 J=1,3
      F(J)=F0(J)
      DO 20 I=1,N
      DO 20 J=1,3
      F(J)=F(J)+FP(J,I)
      DO 30 J=1,3
      SUM(J)=0.0
      DO 40 I=1,N
      DO 50 J=1,3
      WV(J)=RM(J,I)+Q(J,I)
      CALL CROSS(WV,FP(1,I),WV1)
      DO 60 J=1,3
      SUM(J)=SUM(J)+WV1(J)
      DO 70 J=1,3
      JP3=J+3
      F(JP3)=TAU0(J)+SUM(J)
      L=1
      DO 80 I=1,N
      DO 85 J=1,3
      WV2(L)=FP(J,I)
      L=L+1
      CONTINUE
      DO 90 I=1,NT
      IP6=I+6
      F(IP6)=0.0
      DO 95 L=1,N3
      F(IP6)=F(IP6)+PHI(L,I)*WV2(L)

```


LEVEL 2.3.0 (JUNE 78)	EXTF	OS/360 FORTRAN H EXTENDED	DATE 82.246/12.21.52	PAGE 2
-----------------------	------	---------------------------	----------------------	--------

ISN 0040	90	CONTINUE	00058700
ISN 0041		RETURN	00058800
ISN 0042		END	00058900


```

*OPTIONS IN EFFECT*NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTODBL(NONE)
*OPTIONS IN EFFECT*SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTHT NOXREF NOALC NOANSF TERM IBM FLAG(I)
*STATISTICS* SOURCE STATEMENTS = 41, PROGRAM SIZE = 2734, SUBPROGRAM NAME = EXTF
*STATISTICS* NO DIAGNOSTICS GENERATED
***** END OF COMPILATION *****
280K BYTES OF CORE NOT USED

```

REQUESTED OPTIONS: NOOBJ,TERM,,NOXREF,,NOMAP,,,NAME(MAIN),AD(NONE),OPT(0),,,FLAG(I),SIZE(384K),LC(60),

OPTIONS IN EFFECT: NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)
SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

```

ISN 0002      SUBROUTINE NLKT                                00059000
C                                                     00059100
C THIS SUBROUTINE CALCULATES THE NON-LINEAR KINEMATIC TERMS IN THE 00059200
C MOTION EQUATIONS. THESE TERMS ARE ASSEMBLED INTO THE VECTOR "U". 00059300
C                                                     00059400
C                                                     00059500
ISN 0003      IMPLICIT REAL*8(A-H,O-Z)                    00059600
ISN 0004      REAL*8 MASS,MCO,MRM,MQ,MCI,MOS,I0            00059700
ISN 0005      DIMENSION MASS(50),MCO(3),MRM(3,50),CMAT(3,3),MQ(3,50),MCI(3), 00059800
1 QOOT(3,50),UVW(3),OMEGA(3),UT(3),UR(3),UV(150),U(156),MV1(3), 00059900
2 WV2(3),WV3(3),WV4(3),WM1(3,3),Q(3,50),R(3),THETA(3),MOS(3), 00060000
3 IO(3,3),S(3),WS(150),PHI(150,150) 00060100
ISN 0006      COMMON /CONST/ TH,MASS,MCO,MRM,CMAT,N,N3,N3P6,HT,NTP6,NO 00060200
ISN 0007      COMMON /TDEPV/ MQ,MCI 00060300
ISN 0008      COMMON /STATE/ R,THETA,Q,UVW,OMEGA,QOOT 00060400
ISN 0009      COMMON /FICFRC/ U 00060500
ISN 0010      COMMON /RIGID/ MOS,I0,S 00060600
ISN 0011      COMMON /MODES/ PHI,WS 00060700
ISN 0012      EQUIVALENCE(U(1),UT(1)),(U(4),UR(1)) 00060800
C                                                     00060900
C CALCULATE DEFORMATION INDEPENDENT TERMS 00061000
C                                                     00061100
ISN 0013      CALL CROSS(UVW,OMEGA,MV1) 00061200
ISN 0014      SUM=OMEGA(1)*MCO(1)+OMEGA(2)*MCO(2)+OMEGA(3)*MCO(3) 00061300
ISN 0015      OMS=OMEGA(1)**2+OMEGA(2)**2+OMEGA(3)**2 00061400
ISN 0016      DO 20 J=1,3 00061500
ISN 0017      UT(J)=TH*MV1(J)-SUM*OMEGA(J)+OMS*MCO(J) 00061600
ISN 0018      CALL CROSS(MCO,MV1,MV2) 00061700
ISN 0019      DO 30 J=1,3 00061800
ISN 0020      WV3(J)=CMAT(J,1)*OMEGA(1)+CMAT(J,2)*OMEGA(2)+CMAT(J,3)*OMEGA(3) 00061900
ISN 0021      CALL CROSS(OMEGA,WV3,WV4) 00062000
ISN 0022      DO 40 J=1,3 00062100
ISN 0023      UR(J)=WV2(J)+WV4(J) 00062200
ISN 0024      SUM=OMEGA(1)*S(1)+OMEGA(2)*S(2)+OMEGA(3)*S(3) 00062300
ISN 0025      CALL CROSS(MOS,OMEGA,MV2) 00062400
ISN 0026      DO 42 J=1,3 00062500
ISN 0027      WV3(J)=IO(J,1)*OMEGA(1)+IO(J,2)*OMEGA(2)+IO(J,3)*OMEGA(3) 00062600
ISN 0028      CALL CROSS(OMEGA,WV3,WV4) 00062700
ISN 0029      DO 44 J=1,3 00062800
ISN 0030      UR(J)=UR(J)-SUM*WV2(J)-WV4(J) 00062900
ISN 0031      DO 50 I=1,N 00063000
ISN 0032      L=3*I-2 00063100
ISN 0033      DO 55 J=1,3 00063200
ISN 0034      WV2(J)=MASS(I)*WV1(J) 00063300
ISN 0035      SUM=OMEGA(1)*MRM(1,I)+OMEGA(2)*MRM(2,I)+OMEGA(3)*MRM(3,I) 00063400
ISN 0036      DO 60 J=1,3 00063500
ISN 0037      WV3(J)=SUM*OMEGA(J) 00063600
ISN 0038      WV4(J)=OMS*MRM(J,I) 00063700
ISN 0039      LL=L 00063800
ISN 0040      DO 70 J=1,3 00063900
ISN 0041      UV(LL)=WV2(J)-WV3(J)+WV4(J) 00064000
ISN 0042      LL=LL+1 00064100
ISN 0043      CONTINUE 00064200
ISN 0044      DO 74 I=1,NT

```

```

ISN 0045      IP6=I+6                      00064300
ISN 0046      U(IP6)=0.0                  00064400
ISN 0047      DO 74 L=1,N3                00064500
ISN 0048      U(IP6)=U(IP6)+PHI(L,I)*UV(L) 00064600
74           00064700
C           00064800
C CALCULATE FIRST ORDER DEFORMATION DEPENDENT TERMS 00064900
C           00065000
ISN 0049      DO 80 J=1,3                 00065100
ISN 0050      80   WV2(J)=0.0              00065200
ISN 0051      DO 90 I=1,N                 00065300
ISN 0052      DO 95 J=1,3                 00065400
ISN 0053      95   WV2(J)=WV2(J)+MASS(I)*QDOT(J,I) 00065500
ISN 0054      90   CONTINUE                00065600
ISN 0055      CALL CROSS(OMEGA,WV2,WV3)    00065700
ISN 0056      SUM=OMEGA(1)*MC1(1)+OMEGA(2)*MC1(2)+OMEGA(3)*MC1(3) 00065800
ISN 0057      DO 100 J=1,3                00065900
ISN 0058      100  UT(J)=UT(J)-2.0*WV3(J)-SUM*OMEGA(J)+OMS*MC1(J) 00066000
ISN 0059      CALL CROSS(MC1,WV1,WV2)      00066100
ISN 0060      DO 110 I=1,3                00066200
ISN 0061      DO 110 J=1,3                00066300
ISN 0062      110  WM1(I,J)=0.0            00066400
ISN 0063      DO 120 I=1,N                 00066500
ISN 0064      DO 125 J=1,3                 00066600
ISN 0065      125  WM1(J,J)=WM1(J,J)+2.0*MRM(J,I)*Q(J,I) 00066700
ISN 0066      WM1(1,2)=WM1(1,2)+MRM(1,I)*Q(2,I)+MRM(2,I)*Q(1,I) 00066800
ISN 0067      WM1(1,3)=WM1(1,3)+MRM(1,I)*Q(3,I)+MRM(3,I)*Q(1,I) 00066900
ISN 0068      WM1(2,3)=WM1(2,3)+MRM(2,I)*Q(3,I)+MRM(3,I)*Q(2,I) 00067000
ISN 0069      120  CONTINUE                00067100
ISN 0070      DO 130 I=1,3                00067200
ISN 0071      DO 130 J=1,3                00067300
ISN 0072      IF(I.LE. J) GO TO 130        00067400
ISN 0073      WM1(I,J)=WM1(J,I)            00067500
ISN 0074      130  CONTINUE                00067600
ISN 0075      DO 140 J=1,3                 00067700
ISN 0076      140  WV3(J)=WM1(J,1)*OMEGA(1)+WM1(J,2)*OMEGA(2)+WM1(J,3)*OMEGA(3) 00067800
ISN 0077      CALL CROSS(OMEGA,WV3,WV4)    00067900
ISN 0078      SUM=0.0                      00068000
ISN 0079      DO 150 I=1,N                 00068100
ISN 0080      150  SUM=SUM+MRM(1,I)*QDOT(1,I)+MRM(2,I)*QDOT(2,I)+MRM(3,I)*QDOT(3,I) 00068200
ISN 0081      DO 160 J=1,3                 00068300
ISN 0082      160  WV1(J)=SUM*OMEGA(J)      00068400
ISN 0083      DO 170 J=1,3                 00068500
ISN 0084      170  WV3(J)=0.0              00068600
ISN 0085      DO 180 I=1,N                 00068700
ISN 0086      180  SUM=MRM(1,I)*OMEGA(1)+MRM(2,I)*OMEGA(2)+MRM(3,I)*OMEGA(3) 00068800
ISN 0087      DO 190 J=1,3                 00068900
ISN 0088      190  WV3(J)=WV3(J)+SUM*QDOT(J,I) 00069000
ISN 0089      180  CONTINUE                00069100
ISN 0090      DO 200 J=1,3                 00069200
ISN 0091      200  UR(J)=UR(J)+WV2(J)+WV4(J)+2.*(WV3(J)-WV1(J)) 00069300
ISN 0092      DO 300 I=1,N                 00069400
ISN 0093      L=3*I-2                      00069500
ISN 0094      CALL CROSS(OMEGA,QDOT(1,I),WV1) 00069600
ISN 0095      DO 310 J=1,3                 00069700
ISN 0096      310  WV2(J)=MASS(I)*WV1(J)    00069800
ISN 0097      SUM=OMEGA(1)*MQ(1,I)+OMEGA(2)*MQ(2,I)+OMEGA(3)*MQ(3,I) 00069900
ISN 0098      DO 320 J=1,3                 00070000
ISN 0099      320  WV3(J)=SUM*OMEGA(J)
ISN 0100

```

LEVEL 2.3.0 (JUNE 78) NLKT OS/360 FORTRAN H EXTENDED DATE 82.246/12.21.54 PAGE 3

ISN 0101	320	WV4(J)=OM5*MQ(J,I)	00070100
ISN 0102		LL=L	00070200
ISN 0103		DO 330 J=1,3	00070300
ISN 0104		UV(LL)=UV(LL)-2.*WV2(J)-WV3(J)+WV4(J)	00070400
ISN 0105	330	LL=LL+1	00070500
ISN 0106	300	CONTINUE	00070600
ISN 0107		DO 307 I=1,NT	00070700
ISN 0108		IP6=I+6	00070800
ISN 0109		U(IP6)=0.0	00070900
ISN 0110		DO 307 L=1,N3	00071000
ISN 0111	307	U(IP6)=U(IP6)+PHI(L,I)*UV(L)	00071100
	C		00071200
	C	CALCULATE SECOND ORDER DEFORMATION DEPENDENT TERMS	00071300
	C		00071400
ISN 0112		DO 311 I=1,3	00071500
ISN 0113		DO 311 J=1,3	00071600
ISN 0114	311	WM1(I,J)=0.0	00071700
ISN 0115		DO 321 I=1,N	00071800
ISN 0116		DO 331 J=1,3	00071900
ISN 0117	331	WM1(J,J)=WM1(J,J)+MQ(J,I)*MQ(J,I)	00072000
ISN 0118		WM1(1,2)=WM1(1,2)+MQ(1,I)*MQ(2,I)	00072100
ISN 0119		WM1(1,3)=WM1(1,3)+MQ(1,I)*MQ(3,I)	00072200
ISN 0120		WM1(2,3)=WM1(2,3)+MQ(2,I)*MQ(3,I)	00072300
ISN 0121	321	CONTINUE	00072400
ISN 0122		DO 340 I=1,3	00072500
ISN 0123		DO 340 J=1,3	00072600
ISN 0124		IF(I .LE. J) GO TO 340	00072700
ISN 0126		WM1(I,J)=WM1(J,I)	00072800
ISN 0127	340	CONTINUE	00072900
ISN 0128		DO 350 J=1,3	00073000
ISN 0129	350	WV1(J)=WM1(J,1)*OMEGA(1)+WM1(J,2)*OMEGA(2)+WM1(J,3)*OMEGA(3)	00073100
ISN 0130		CALL CROSS(OMEGA,WV1,WV2)	00073200
ISN 0131		SUM=0.0	00073300
ISN 0132		DO 360 I=1,N	00073400
ISN 0133	360	SUM=SUM+MQ(1,I)*QDOT(1,I)+MQ(2,I)*QDOT(2,I)+MQ(3,I)*QDOT(3,I)	00073500
ISN 0134		DO 370 J=1,3	00073600
ISN 0135		WV3(J)=SUM*OMEGA(J)	00073700
ISN 0136	370	WV4(J)=0.0	00073800
ISN 0137		DO 380 I=1,N	00073900
ISN 0138		SUM=OMEGA(1)*MQ(1,I)+OMEGA(2)*MQ(2,I)+OMEGA(3)*MQ(3,I)	00074000
ISN 0139		DO 385 J=1,3	00074100
ISN 0140	385	WV4(J)=WV4(J)+SUM*QDOT(J,I)	00074200
ISN 0141	380	CONTINUE	00074300
ISN 0142		DO 390 J=1,3	00074400
ISN 0143	390	UR(J)=UR(J)+WV2(J)+2.*(WV4(J)-WV3(J))	00074500
ISN 0144		RETURN	00074600
ISN 0145		END	00074700

*OPTIONS IN EFFECT*NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTODBL(NONE)
 *OPTIONS IN EFFECT*SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT 60STMT NOXREF NOALC NOANSF TERM IBM FLAG(I)
 STATISTICS SOURCE STATEMENTS = 144, PROGRAM SIZE = 8226, SUBPROGRAM NAME = NLKT
 STATISTICS NO DIAGNOSTICS GENERATED
 ***** END OF COMPILATION *****
 244K BYTES OF CORE NOT USED

REQUESTED OPTIONS: NOOBJ,TERM,,NOXREF,,NOMAP,,,NAME(MAIN),AD(NONE),OPT(0),,,FLAG(I),SIZE(384K),LC(60),

OPTIONS IN EFFECT: NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)
SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

```

ISN 0002      SUBROUTINE SOLVE                                00074800
C                                                     00074900
C THIS SUBROUTINE ASSEMBLES THE EQUATIONS OF MOTION IN FIRST ORDER 00075000
C FORM AND SOLVES THE SET OF SIMULTANEOUS DIFFERENTIAL EQUATIONS 00075100
C (SIZE 6*N+12) 00075200
ISN 0003      IMPLICIT REAL*8(A-H,O-Z) 00075300
ISN 0004      REAL*8 MASS,MC0,MRM,INERT1 00075400
ISN 0005      DIMENSION MASS(50),MC0(3),MRM(3,50),CMAT(3,3),A(156,156),A12(3,3),00075500
1 INERT1(3,3),A23(3,150),Q(3,50),QDOT(3,50),UVH(3),OMEGA(3), 00075600
2 R(3),THETA(3),F(156),U(156),R14(3,3),GAMA(3,3), 00075700
3 B(156),YDOT(312),Y(312),WV1(3),WV2(3),WV3(156), 00075800
4 QV(150),QDOTV(150),AV(12246) 00075900
ISN 0006      EQUIVALENCE (QV(1),Q(1,1)),(QDOTV(1),QDOT(1,1)) 00076000
ISN 0007      COMMON /CONST/ TH,MASS,MC0,MRM,CMAT,N,N3,N3P6,NT,NTP6,NO 00076100
ISN 0008      COMMON /AMAT/ A,A12,INERT1,A23 00076200
ISN 0009      COMMON /STATE/ R,THETA,Q,UVH,OMEGA,QDOT 00076300
ISN 0010      COMMON /FORCE/ F 00076400
ISN 0011      COMMON /FICFRC/ U 00076500
ISN 0012      COMMON /TIME/ DT,TSTOP,DTP,DTG 00076600
ISN 0013      COMMON /MODES/ PHI(150,150),WS(150) 00076700
ISN 0014      COMMON /MODCO/ ETA(150),ETAD(150) 00076800
ISN 0015      DATA GAMA/6*0.0,1.0/,IPASS/0/ 00076900
C 00077000
C CALCULATE R14 -TRANSFORMATION FROM BODY FRAME TO INERTIAL FRAME 00077100
C 00077200
ISN 0016      S1=DSIN(THETA(1)) 00077300
ISN 0017      C1=DCOS(THETA(1)) 00077400
ISN 0018      S2=DSIN(THETA(2)) 00077500
ISN 0019      C2=DCOS(THETA(2)) 00077600
ISN 0020      S3=DSIN(THETA(3)) 00077700
ISN 0021      C3=DCOS(THETA(3)) 00077800
ISN 0022      R14(1,1)=C2*MC3 00077900
ISN 0023      R14(1,2)=-C2*S3 00078000
ISN 0024      R14(1,3)=S2 00078100
ISN 0025      R14(2,1)=C1*S3+S1*S2*MC3 00078200
ISN 0026      R14(2,2)=C1*MC3-S1*S2*S3 00078300
ISN 0027      R14(2,3)=-S1*C2 00078400
ISN 0028      R14(3,1)=S1*S3-C1*S2*MC3 00078500
ISN 0029      R14(3,2)=S1*MC3+C1*S2*S3 00078600
ISN 0030      R14(3,3)=C1*C2 00078700
C 00078800
C CALCULATE "GAMA" - TRANSFORMS ANGULAR VELOCITY TO ATTITUDE RATES 00078900
C 00079000
ISN 0031      GAMA(1,1)=C3/C2 00079100
ISN 0032      GAMA(1,2)=-S3/C2 00079200
ISN 0033      GAMA(2,1)=S3 00079300
ISN 0034      GAMA(2,2)=C3 00079400
ISN 0035      T2=S2/C2 00079500
ISN 0036      GAMA(3,1)=-C3*T2 00079600
ISN 0037      GAMA(3,2)=S3*T2 00079700
ISN 0038      DO 30 I=1,6 00079800
ISN 0039      30 B(I)=F(I)+U(I) 00079900
ISN 0040      DO 40 I=1,NT 00080000

```

```

ISN 0041      IP6=I+6
ISN 0042      40  B(IP6)=F(IP6)+U(IP6)-WS(I)*ETA(I)
C
C      STORE UPPER TRIANGLE OF "A" IN "AV"
C
ISN 0043      L=1
ISN 0044      DO 300 J=1,NTP6
ISN 0045      DO 300 I=1,J
ISN 0046      AV(L)=A(I,J)
ISN 0047      L=L+1
ISN 0048      300  CONTINUE
ISN 0049      CALL LEQT1P(AV,1,NTP6,B,156,0,D1,D2,IER)
ISN 0050      IF(IER .EQ. 0) GO TO 60
ISN 0052      WRITE(6,61) IER
ISN 0053      STOP
ISN 0054      61  FORMAT(1H0,5X,'ERROR DETECTED BY INSL LIBRARY ROUTINE "LEQT1P"
C      1ERROR CODE=',I3)
ISN 0055      60  IF(IPASS .EQ. 1) GO TO 105
ISN 0057      IPASS=1
C
C      SET INITIAL VALUE OF "Y"
C
ISN 0058      DO 70 I=1,3
ISN 0059      IP3=I+3
ISN 0060      Y(I)=R(I)
ISN 0061      70  Y(IP3)=THETA(I)
ISN 0062      DO 80 I=1,NT
ISN 0063      80  Y(I+6)=ETA(I)
ISN 0064      DO 90 I=1,3
ISN 0065      L=NTP6+I
ISN 0066      LL=L+3
ISN 0067      Y(L)=UVH(I)
ISN 0068      90  Y(LL)=OMEGA(I)
ISN 0069      DO 100 I=1,NT
ISN 0070      L=12+NT+I
ISN 0071      100 Y(L)=ETAD(I)
C
C      SET UP "YDOT"
C
ISN 0072      105 DO 110 I=1,3
ISN 0073      WV1(I)=R14(I,1)*UVH(1)+R14(I,2)*UVH(2)+R14(I,3)*UVH(3)
ISN 0074      110 WV2(I)=GAMA(I,1)*OMEGA(1)+GAMA(I,2)*OMEGA(2)+GAMA(I,3)*OMEGA(3)
ISN 0075      DO 120 I=1,3
ISN 0076      YDOT(I)=WV1(I)
ISN 0077      120 YDOT(I+3)=WV2(I)
ISN 0078      DO 130 I=1,NT
ISN 0079      130 YDOT(6+I)=ETAD(I)
ISN 0080      DO 140 I=1,NTP6
ISN 0081      140 YDOT(NTP6+I)=B(I)
C
C      UPDATE VARIABLES IN STATE VECTOR
C
ISN 0082      CALL ODES1V(NO,Y,YDOT,DT)
ISN 0083      DO 150 I=1,3
ISN 0084      R(I)=Y(I)
ISN 0085      150 THETA(I)=Y(I+3)
ISN 0086      DO 160 I=1,NT
ISN 0087      160 ETA(I)=Y(6+I)

```

ISN 0088		DO 170 I=1,3	00085900
ISN 0089		L=NTP6+I	00086000
ISN 0090		LL=L+3	00086100
ISN 0091		UVN(I)=Y(L)	00086200
ISN 0092	170	OMEGA(I)=Y(LL)	00086300
ISN 0093		DO 180 I=1,NT	00086400
ISN 0094	180	ETAD(I)=Y(NT+12+I)	00086500
	C		00086600
	C	COMPUTE NEW VALUES FOR "Q" AND "QDOT"	00086700
	C		00086800
ISN 0095		DO 200 I=1,N3	00086900
ISN 0096		QV(I)=0.0	00087000
ISN 0097		QDOTV(I)=0.0	00087100
ISN 0098		DO 220 L=1,NT	00087200
ISN 0099		QV(I)=QV(I)+PHI(I,L)*ETA(L)	00087300
ISN 0100	220	QDOTV(I)=QDOTV(I)+PHI(I,L)*ETAD(L)	00087400
ISN 0101	200	CONTINUE	00087500
ISN 0102		RETURN	00087600
ISN 0103		END	00087700

*OPTIONS IN EFFECT*NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)

*OPTIONS IN EFFECT*SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

STATISTICS SOURCE STATEMENTS = 102, PROGRAM SIZE = 107808, SUBPROGRAM NAME = SOLVE

STATISTICS NO DIAGNOSTICS GENERATED

***** END OF COMPILATION *****

256K BYTES OF CORE NOT USED

REQUESTED OPTIONS: NOOBJ,TERM,,NOXREF,,NOMAP,,,NAME(MAIN),AD(NONE),OPT(0),,,FLAG(I),SIZE(384K),LC(60),

OPTIONS IN EFFECT: NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)
SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

ISN 0002		SUBROUTINE ODESIV(N,Y,DERIV,H)	00087800
ISN 0003		IMPLICIT REAL*8(A-H,O-Z)	00087900
ISN 0004		DIMENSION DERIV(N),Y(N),DERIVO(312),BD1(312,2),BD2(312,2),BD3(312)	00088000
ISN 0005		DATA INTF/1/,C1/0.0/,C2/0.0/,C3/0./	00088100
	C		00088200
	C	THIS SUBROUTINE INTEGRATES THE FIRST ORDER SYSTEM OF ORDINARY	00088300
	C	DIFFERENTIAL EQUATIONS "DY/DT=DERIV" BY THE ADAMS METHOD	00088400
	C	USING THIRD ORDER DIFFERENCES.	00088500
	C	N- SIZE OF SYSTEM	00088600
	C	Y- VECTOR OF INITIAL VALUES ON INPUT. "Y" IS OVERWRITTEN	00088700
	C	WITH THE NEW SOLUTION	00088800
	C	H- STEP SIZE	00088900
	C		00089000
	C		00089100
ISN 0006		IF(N .LE. 312) GO TO 10	00089200
ISN 0008		WRITE(6,12) N	00089300
ISN 0009		STOP	00089400
ISN 0010	12	FORMAT(1H0.5X,'ERROR IN SUBROUTINE **ODESLV** CALLED WITH STATE	00089500
		1SIZE =',I3,' EXCEEDS DIMENSION SIZE OF ARRAYS')	00089600
ISN 0011	10	GO TO(1000,2000,3000,4000),INTF	00089700
	C		00089800
	C	FIRST CALL TO ROUTINE - EULER INTEGRATION	00089900
	C		00090000
ISN 0012	1000	DO 20 I=1,N	00090100
ISN 0013	20	DERIVO(I)=DERIV(I)	00090200
ISN 0014		INTF=2	00090300
ISN 0015		GO TO 5000	00090400
	C		00090500
	C	SECOND CALL TO ROUTINE - FIRST ORDER DIFFERENCES	00090600
	C		00090700
ISN 0016	2000	DO 30 I=1,N	00090800
ISN 0017		BD1(I,1)=DERIV(I)-DERIVO(I)	00090900
ISN 0018		BD1(I,2)=BD1(I,1)	00091000
ISN 0019	30	DERIVO(I)=DERIV(I)	00091100
ISN 0020		C1=.5	00091200
ISN 0021		INTF=3	00091300
ISN 0022		GO TO 5000	00091400
	C		00091500
	C	THIRD CALL TO ROUTINE - SECOND ORDER DIFFERENCES	00091600
	C		00091700
ISN 0023	3000	DO 40 I=1,N	00091800
ISN 0024		BD1(I,2)=DERIV(I)-DERIVO(I)	00091900
ISN 0025		BD2(I,1)=BD1(I,2)-BD1(I,1)	00092000
ISN 0026		BD2(I,2)=BD2(I,1)	00092100
ISN 0027		DERIVO(I)=DERIV(I)	00092200
ISN 0028	40	BD1(I,1)=BD1(I,2)	00092300
ISN 0029		INTF=4	00092400
ISN 0030		C2=5.0/12.0	00092500
ISN 0031		GO TO 5000	00092600
	C		00092700
	C	ADAMS METHOD WITH 3RD ORDER DIFFERENCES.	00092800
	C		00092900
ISN 0032	4000	DO 50 I=1,N	00093000
ISN 0033		BD1(I,2)=DERIV(I)-DERIVO(I)	

ISN 0034		BD2(I,2)=BD1(I,2)-BD1(I,1)	00093100
ISN 0035		BD3(I)=BD2(I,2)-BD2(I,1)	00093200
ISN 0036		DERIVO(I)=DERIV(I)	00093300
ISN 0037		BD1(I,1)=BD1(I,2)	00093400
ISN 0038	50	BD2(I,1)=BD2(I,2)	00093500
ISN 0039		C3=3.0/8.0	00093600
ISN 0040		GO TO 5000	00093700
	C		00093800
	C	UPDATE VECTOR 'Y'	00093900
	C		00094000
ISN 0041	5000	DO 60 I=1,N	00094100
ISN 0042	60	Y(I)=Y(I)+H*(DERIV(I)+C1*BD1(I,2)+C2*BD2(I,2)+C3*BD3(I))	00094200
ISN 0043		RETURN	00094300
ISN 0044		END	00094400

*OPTIONS IN EFFECT*NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)

*OPTIONS IN EFFECT*SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

STATISTICS SOURCE STATEMENTS = 43, PROGRAM SIZE = 16838, SUBPROGRAM NAME =ODESLV

STATISTICS NO DIAGNOSTICS GENERATED

***** END OF COMPILATION *****

276K BYTES OF CORE NOT USED

REQUESTED OPTIONS: NOOBJ,TERM,,NOXREF,,NOMAP,,,NAME(MAIN),AD(NONE),OPT(0),,,FLAG(I),SIZE(384K),LC(60),

OPTIONS IN EFFECT: NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)
SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTHT NOXREF NOALC NOANSF TERM IBM FLAG(I)

ISN 0002	SUBROUTINE PRINT(T,F0,TAU0,FP)	00094500
ISN 0003	IMPLICIT REAL*8(A-H,O-Z)	00094600
ISN 0004	REAL*8 MASS,MC0,MRM	00094700
ISN 0005	REAL*4 TD(3),OD(3),PLTDAT	00094800
ISN 0006	DIMENSION R(3),THETA(3),Q(3,50),UVW(3),OMEGA(3),QDOT(3,50),	00094900
	1 MC0(3),MRM(3,50),CHAT(3,3),MASS(50),F0(3),TAU0(3),FP(3,50),	00095000
	2 IPLOT(42),PLTDAT(100,20)	00095100
ISN 0007	COMMON /STATE/R,THETA,Q,UVW,OMEGA,QDOT	00095200
ISN 0008	COMMON /CONST/ TH,MASS,MC0,MRM,CHAT,N,N3,N3P6,NT,NTP6,NO	00095300
ISN 0009	COMMON /PLOTT/ PLTDAT,IPLOT,NP	00095400
ISN 0010	WRITE(6,100) T	00095500
ISN 0011	WRITE(6,200) F0,TAU0	00095600
ISN 0012	DO 201 I=1,N	00095700
ISN 0013	201 WRITE(6,202) I,(FP(J,I),J=1,3)	00095800
ISN 0014	DO 10 I=1,3	00095900
ISN 0015	TD(I)=57.29578*THETA(I)	00096000
ISN 0016	10 OD(I)=57.29578*OMEGA(I)	00096100
ISN 0017	WRITE(6,110) R,TD,UVW,OD	00096200
ISN 0018	DO 20 I=1,N	00096300
ISN 0019	20 WRITE(6,120) I,(Q(J,I),J=1,3)	00096400
ISN 0020	DO 30 I=1,N	00096500
ISN 0021	30 WRITE(6,130) I,(QDOT(J,I),J=1,3)	00096600
ISN 0022	100 FORMAT(1H0,///,5X,'TIME=',F10.3,' SEC')	00096700
ISN 0023	110 FORMAT(1H ,8X,'R=',3(2X,1PE11.4),' FT',/,4X,'THETA=',	00096800
	1 3(2X,1PE11.4),' DEG',/,6X,'UVW=',3(2X,1PE11.4),' FT/SEC',/,	00096900
	2 4X,'OMEGA=',3(2X,1PE11.4),' DEG/SEC',/)	00097000
ISN 0024	120 FORMAT(1H ,5X,'Q',I3,'=',3(2X,1PE11.4),' FT')	00097100
ISN 0025	130 FORMAT(1H ,2X,'QDOT',I3,'=',3(2X,1PE11.4),' FT/SEC')	00097200
ISN 0026	200 FORMAT(1H0,8X,'F0=',3(2X,1PE11.4),' LB',/,	00097300
	1 6X,'TAU0=',3(2X,1PE11.4),' FT LB')	00097400
ISN 0027	202 FORMAT(1H ,4X,'F',I3,'=',3(2X,1PE11.4),' LB')	00097500
ISN 0028	RETURN	00097600
ISN 0029	ENTRY GRAF(T)	00097700
ISN 0030	IF(IPLOT(1) .EQ. 0) GO TO 203	00097800
	C	00097900
	C STORE VARIABLES FOR PLOTTING	00098000
	C	00098100
ISN 0032	NP=NP+1	00098200
ISN 0033	IF(NP .GT. 100) GO TO 203	00098300
ISN 0035	PLTDAT(NP,1)=T	00098400
ISN 0036	DO 300 I=1,42	00098500
ISN 0037	NY=IPLOT(I)	00098600
ISN 0038	IF(NY .EQ. 0) GO TO 203	00098700
ISN 0040	IF(NY .GT. 3) GO TO 310	00098800
	C	00098900
	C STORE INERTIAL POSITION	00099000
	C	00099100
ISN 0042	PLTDAT(NP,I+1)=R(NY)	00099200
ISN 0043	GO TO 300	00099300
ISN 0044	310 IF(NY .GT. 6) GO TO 320	00099400
	C	00099500
	C STORE ATTITUDE ANGLES IN DEGREES	00099600
	C	00099700

ISN 0046		PLTDAT(NP,I+1)=57.29578*THETA(NY-3)	00099800
ISN 0047		GO TO 300	00099900
ISN 0048	320	IF(NY .GT. N3P6) GO TO 330	00100000
	C		00100100
	C	STORE DEFORMATION COORDINATES	00100200
	C		00100300
ISN 0050		L=NY-6	00100400
ISN 0051		I1=1+L/3	00100500
ISN 0052		I2=L-3*(I1-1)	00100600
ISN 0053		IF(I2 .NE. 0) GO TO 321	00100700
ISN 0055		I1=I1-1	00100800
ISN 0056		I2=3	00100900
ISN 0057	321	PLTDAT(NP,I+1)=Q(I2,I1)	00101000
ISN 0058		GO TO 300	00101100
ISN 0059	330	IF(NY .GT. (N3P6+3)) GO TO 340	00101200
	C		00101300
	C	STORE TRANSLATIONAL VELOCITY	00101400
	C		00101500
ISN 0061		PLTDAT(NP,I+1)=UVW(NY-N3P6)	00101600
ISN 0062		GO TO 300	00101700
ISN 0063	340	IF(NY .GT. (N3P6+6)) GO TO 350	00101800
	C		00101900
	C	STORE ANGULAR VELOCITY IN DEG./SEC.	00102000
	C		00102100
ISN 0065		PLTDAT(NP,I+1)=57.29578*OMEGA(NY-(N3+9))	00102200
ISN 0066		GO TO 300	00102300
	C		00102400
	C	STORE DEFORMATION RATES	00102500
	C		00102600
ISN 0067	350	L=NY-(N3+12)	00102700
ISN 0068		I1=1+L/3	00102800
ISN 0069		I2=L-3*(I1-1)	00102900
ISN 0070		IF(I2 .NE. 0) GO TO 351	00103000
ISN 0072		I1=I1-1	00103100
ISN 0073		I2=3	00103200
ISN 0074	351	PLTDAT(NP,I+1)=QDOT(I2,I1)	00103300
ISN 0075	300	CONTINUE	00103400
ISN 0076	203	RETURN	00103500
ISN 0077		END	00103600

*OPTIONS IN EFFECT*NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)

*OPTIONS IN EFFECT*SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

STATISTICS SOURCE STATEMENTS = 76, PROGRAM SIZE = 2324, SUBPROGRAM NAME = PRINT

STATISTICS NO DIAGNOSTICS GENERATED

***** END OF COMPILATION *****

268K BYTES OF CORE NOT USED

REQUESTED OPTIONS: NOOBJ,TERM,,NOXREF,,NOMAP,,,NAME(MAIN),AD(NONE),OPT(0),,,FLAG(I),SIZE(384K),LC(60),

OPTIONS IN EFFECT: NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)
SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

ISN 0002		SUBROUTINE PLOT	00103700
ISN 0003		IMPLICIT REAL*8(A-H,O-Z)	00103800
ISN 0004		REAL*4 RAN,PLTDAT	00103900
ISN 0005		DIMENSION IPLOT(42),PLTDAT(100,20),IT(144),RAN(4),IC(10),	00104000
		1 IMAG4(5151)	00104100
ISN 0006		COMMON /PLOT/ PLTDAT,IPLOT,NP	00104200
ISN 0007		DATA IT(1)/0/,RAN/4*0./,IC(1)/1H*/	00104300
	C		00104400
	C	CALCULATE NUMBER OF VARIABLES TO BE PLOTTED	00104500
	C		00104600
			00104700
ISN 0008		NV=0	00104800
ISN 0009		DO 100 I=1,42	00104900
ISN 0010		IF(IPLOT(I) .EQ. 0) GO TO 110	00105000
ISN 0012	100	NV=Nv+1	00105100
ISN 0013	110	IF(NV .EQ. 0) RETURN	00105200
ISN 0015		DO 120 IP=1,NV	00105300
ISN 0016		CALL USPLT(PLTDAT(1,1),PLTDAT(1,IP+1),100,NP,1,1,IT,RAN,IC,1,	00105400
		1 IMAG4,IER)	00105500
ISN 0017	120	CONTINUE	00105600
ISN 0018		RETURN	00105700
ISN 0019		END	

*OPTIONS IN EFFECT*NAME(MAIN) NOOPTIMIZE LINECOUNT(60) SIZE(0384K) AUTOOBL(NONE)

*OPTIONS IN EFFECT*SOURCE EBCDIC NOLIST NODECK NOOBJECT NOMAP NOFORMAT GOSTMT NOXREF NOALC NOANSF TERM IBM FLAG(I)

STATISTICS SOURCE STATEMENTS = 18, PROGRAM SIZE = 21736, SUBPROGRAM NAME = PLOT

STATISTICS NO DIAGNOSTICS GENERATED

***** END OF COMPILATION *****

276K BYTES OF CORE NOT USED

STATISTICS NO DIAGNOSTICS THIS STEP

APPENDIX B

SAMPLE INPUT DATA

This appendix provides an illustrative example of the program NAMELIST input data corresponding to the vehicle in Figure 5. The vector geometry and inertia matrix for that vehicle are

$$\vec{S} = -\frac{b}{2} \underline{i}_4 + \frac{b}{2} \underline{j}_4 + \frac{h}{2} \underline{k}_4$$

$$\vec{r}^1 = -\frac{b}{2} \underline{i}_4 + (b + L) \underline{j}_4 + \frac{h}{2} \underline{k}_4$$

$$\vec{r}^2 = -\frac{b}{2} \underline{i}_4 + (b + 2L) \underline{j}_4 + \frac{h}{2} \underline{k}_4$$

$$\vec{r}^3 = -\frac{b}{2} \underline{i}_4 - L \underline{j}_4 + \frac{h}{2} \underline{k}_4$$

$$\vec{r}^4 = -\frac{b}{2} \underline{i}_4 - 2L \underline{j}_4 + \frac{h}{2} \underline{k}_4$$

$$[I_b] = \frac{m_b}{12} \begin{bmatrix} (b^2 + h^2) & 0 & 0 \\ 0 & (b^2 + h^2) & 0 \\ 0 & 0 & 2b^2 \end{bmatrix}$$

$$\vec{R} = R_x \underline{i}_1 + R_y \underline{j}_1 + R_z \underline{k}_1$$

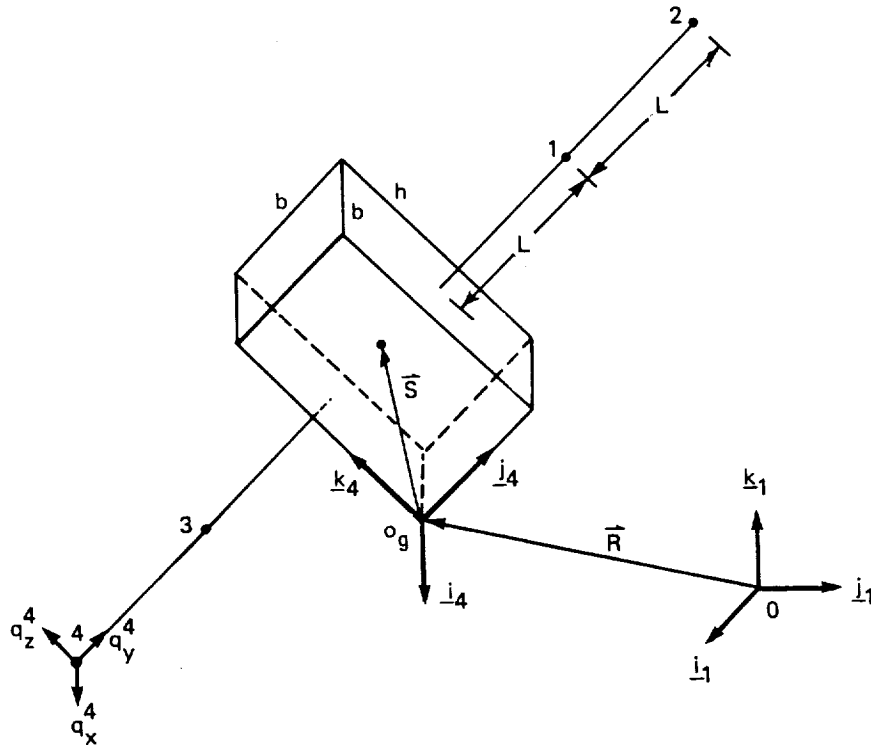


Figure 5. Example vehicle.

The namelist input items given below correspond to the following dimensions and masses

$$m_b = 5 \text{ slugs}$$

$$m_1 = m_3 = 1 \text{ slug}$$

$$m_2 = m_4 = 0.5 \text{ slug}$$

$$b = 1 \text{ ft}$$

$$h = 2 \text{ ft}$$

$$L = 10 \text{ ft}$$

Also two constrained modes are to be used in the simulation. The initial conditions on the kinematic variables are

$$\begin{array}{lll}
 @ t_0 & R_x = 1 \cdot 10^3 \text{ ft} & \theta_1 = 5 \text{ degrees} \\
 & R_y = 2 \cdot 10^3 \text{ ft} & \theta_2 = 20 \text{ degrees} \\
 & R_z = 3 \cdot 10^3 \text{ ft} & \theta_3 = 0 \text{ degrees} \\
 & u = 0 \text{ ft/s} & \omega_1 = 0 \text{ deg/s} \\
 & v = 5 \text{ ft/s} & \omega_2 = 10 \text{ deg/s} \\
 & w = 0 \text{ ft/s} & \omega_3 = 0 \text{ deg/s}
 \end{array}$$

Note that the program in Appendix A sets the initial particle deflections, modal coordinates and the respective time derivatives to zero (see sub-routine INITL).

The numerical integration is to proceed from time = 0 (set internally, see main program) to a final time of 60 seconds using an integration time step of 0.01 second. The print time step is to be 6 seconds and the plot time step 0.6 second.

The following variables are to be plotted versus time: R_y , θ_2 , q_x^4 , q_y^4 , q_z^4 , v , ω_2 .

NAMelist Input Data

```
&INPUT MØ = 5.0, N = 4, MASS = 1.0, 0.5, 1.0, 0.5,  
RM = -0.5, 11.0, 1.0, -0.5, 21.0, 1.0, -0.5, -10.0, 1.0, -0.5,  
      -20.0, 1.0,  
IØ = 2.083, 0.0, 0.0, 0.0, 2.083, 0.0, 0.0, 0.0, 0.833,  
S = -0.5, 0.5, 1.0, NT = 2 &END  
  
&KIN R = 1.E3, 2.E3, 3.E3, THETA = 5.0, 20.0, 0.0,  
UVW = 0.0, 5.0, 0.0, OMEGA = 0.0, 10.0, 0.0 &END  
  
&RUN DT = 0.01, TSTOP = 60.0, DTP = 6.0, DTG = 0.6 &END  
  
&PLT IPLOT = 2, 5, 16, 17, 18, 20, 23 &END
```


LIST OF REFERENCES

1. Hughes, P.C., "A Model for the Attitude Dynamics of CTS with Reference to Attitude Control System Design," Aerospace Engineering and Research Consultants Ltd., Downsview, Ont., AERCOL Report No. 75-14-4, 1975.
2. Likins, P.W., "Analytical Dynamics and Nonrigid Spacecraft Simulation," Technical Report 32-1593, Jet Propulsion Laboratory, Pasadena, CA 1974.
3. Bodley, C.S., A.D. Devers, A.C. Park, and H.P. Frisch, "A Digital Computer Program for the Dynamic Interaction Simulation of Controls and Structures (DISCOS)," Volume I, NASA Technical Paper 1219, 1978.
4. Gates, S.S., "DISCOS Method for Incorporation of Finite-Element Model," Draper Intralab Memo DYN-82-1, January 1982.
5. Likins, P.W., "Dynamics and Control of Flexible Space Vehicles," Technical Report 32-1329, Revision 1, Jet Propulsion Laboratory, Pasadena, CA, 1970.
6. Likins, P.W., "Finite Element Appendage Equations for Hybrid Coordinate Dynamic Analysis," International Journal of Solids and Structures, Volume 8, pp. 709-731, 1972.

7. Storch, J., "Dynamic Equations for Arbitrary Motion of a Flexible Body - Finite Element Idealization," Draper Intralab Memo DYN-82-4, January 1982.
8. IMSL Library Reference Manual, Edition 8, International Mathematical and Statistical Libraries, Inc., Houston, TX, 1980.

